

Low-Swirl Flame Stabilization Method for Lean Premixed Turbulent Flames and Its Adaptation to Heating and Power Equipment

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Presentation at ABMA Mid-Winter Meeting Jan. 18, 2004

Acknowledgement

□ **Sponsors**

- DOE-Basic Energy Sciences, Chemical Sciences
- DOE-Basic Energy Sciences, Laboratory Technology Research
- California Institute of Energy Efficiency/SoCalGas
- DOE-EERE, Office of Industrial Technology
- DOE-EERE, Distributed Energy Resources

□ **Collaborators**

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H. Rieher (Industrial Combustion)

Modes of Gaseous Combustion



Diffusion Flame
controlled by mixing



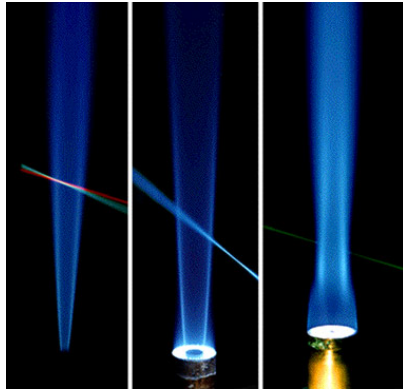
Partially Premixed Flame
two reaction zones



Lean Premixed Flame
wave-like flame front

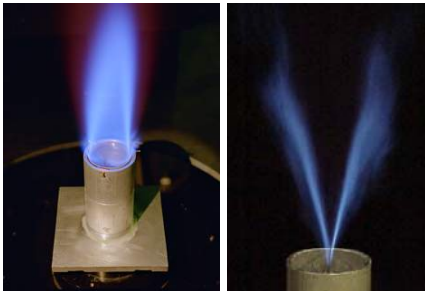
Turbulent Combustion as a Fundamental Research Problem

- No unified theory due to differences in the predominant physical processes of non-premixed and premixed flames



- ***Non-premixed (diffusion) flames***

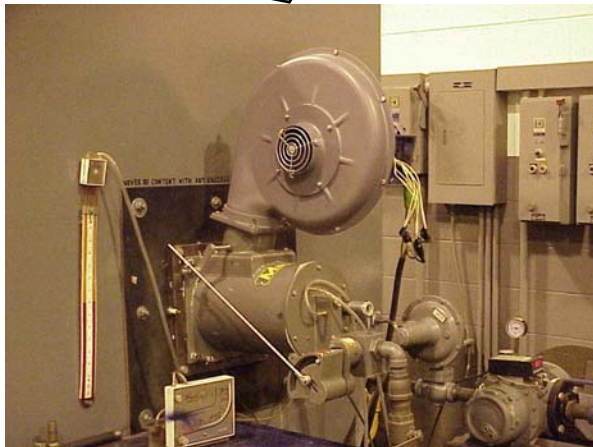
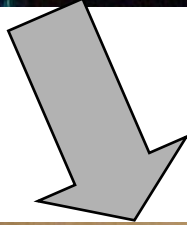
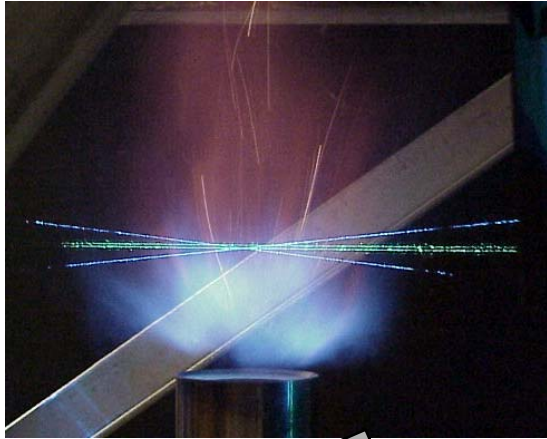
- Turbulent and molecular mixing control combustion rates, efficiency and pollutant formation
 - Reactions occur at stoichiometric contours passive to turbulence
 - Combustion products diffuse into fuel and oxidizer streams
- Reaction rate models expressed in terms of species concentrations



- ***Premixed flames***

- Self propagating flame front separate reactants from products
- Flame front exhibits wave behavior and generates significant feedback to turbulent field through the pressure field
- Reaction rate models expressed in terms of flame speed

LBNL's Basic Research Focuses on Gaseous Premixed Turbulent Flames



□ Theoretical Interest

- Turbulence intensity and sizes of eddies control burning rate, power density and flame stability

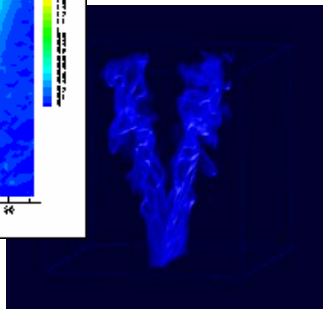
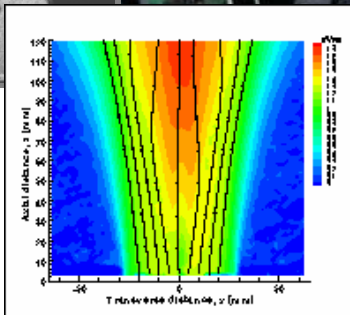
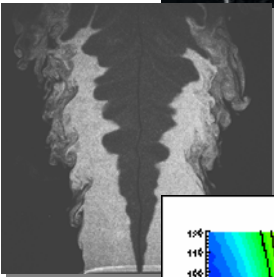
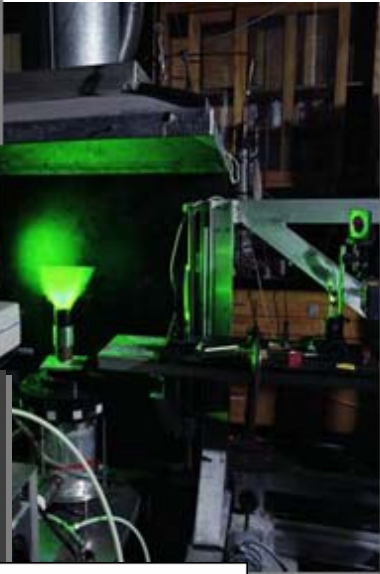
□ Technological Interest

- Reduction of NO_x emissions through lean combustion

□ Programmatic objectives

- Elucidate turbulence/flame interactions processes
- Build an experimental foundation to advance combustion theories & models
- Transfer scientific knowledge to practical use

LBNL's Basic Research Emphasizes Combustion Fluid Mechanics



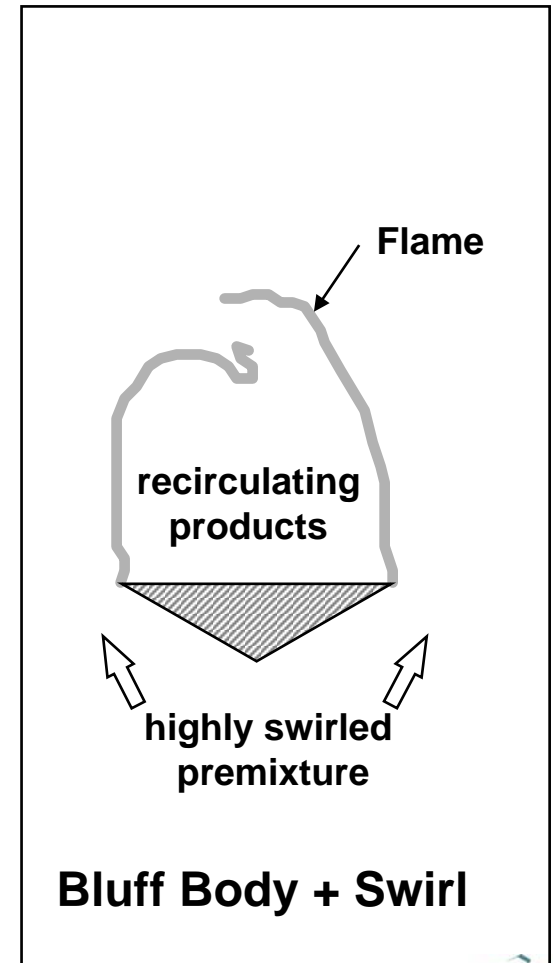
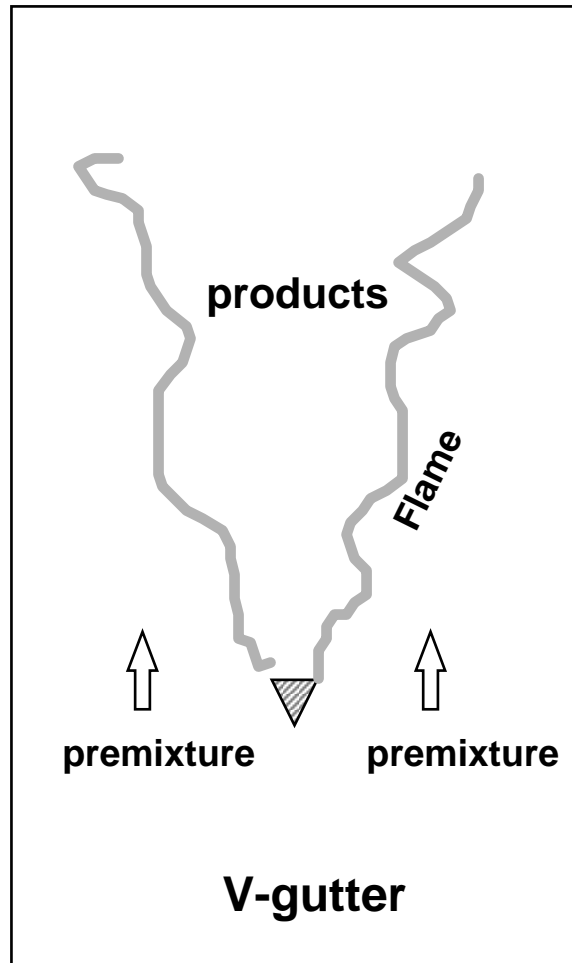
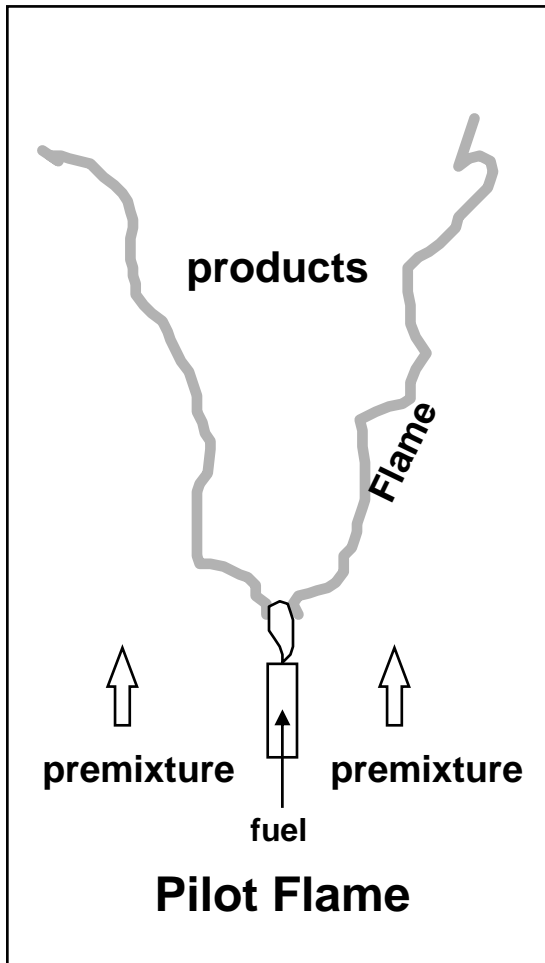
- **Approach:** Laboratory investigations and theoretical development to quantify flame turbulence interactions
 - “clean” experiments to reveal and isolate various processes
 - systematic variation of combustion and turbulence parameters
- **Goal:** Support the development of computational tools suitable for the design of advanced combustion systems

Addressing Problems Relevant to Lean Premixed Combustion Systems

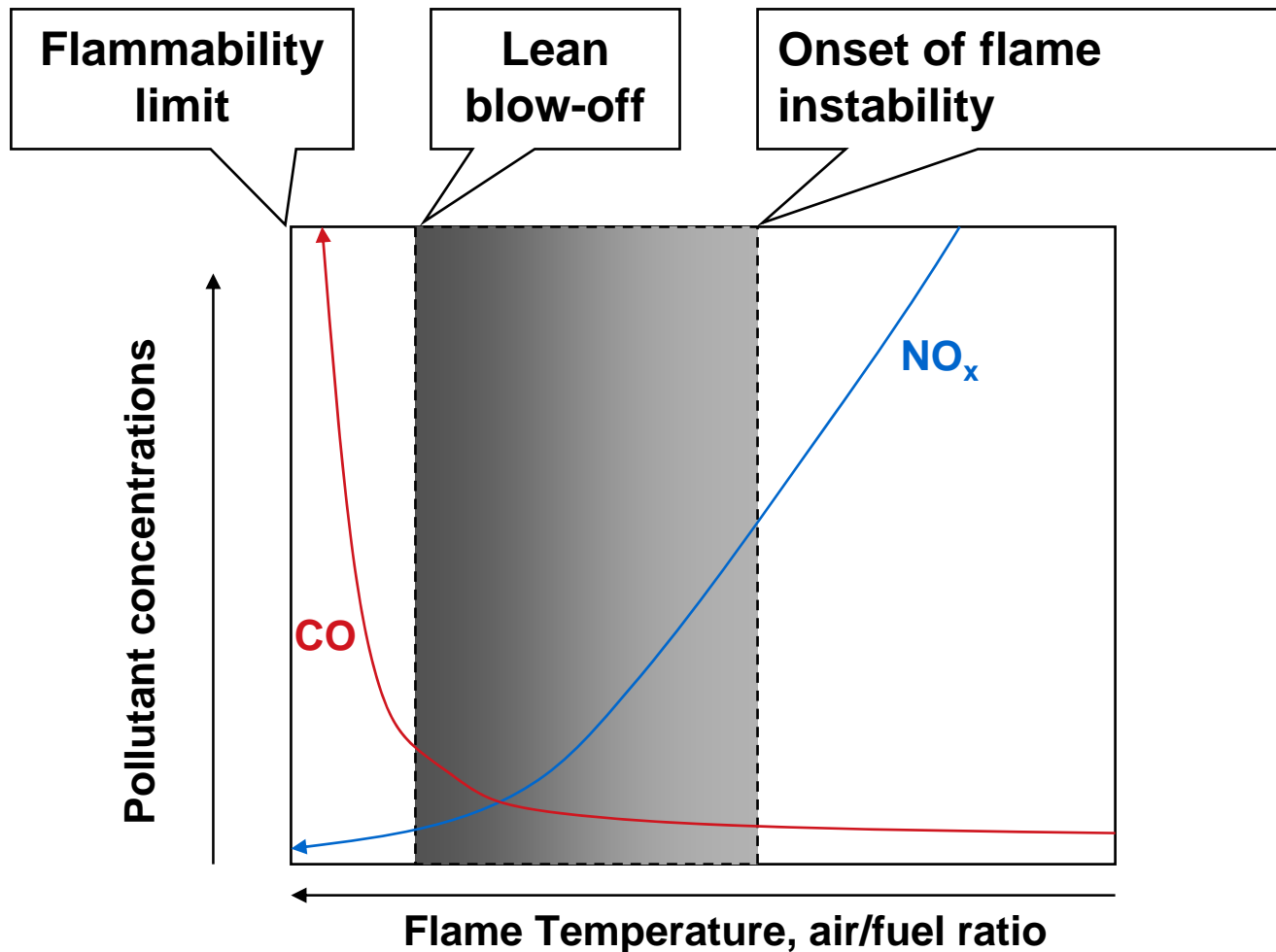
- Incomplete knowledge on flame behavior
 - fast burning, compact & intense flames
 - turbulence effects on emissions
 - flame interaction with combustion chambers
- **Flame holders** dictates performance
 - restrict operating range (5:1 turn-down vs. 10:1 for non-premixed systems)
 - impact fuel flexibility, costs, & durability
- Flame generated flow dynamics
 - noise and vibrations
 - flash-back and blow-out hazards

Conventional Flame Holders

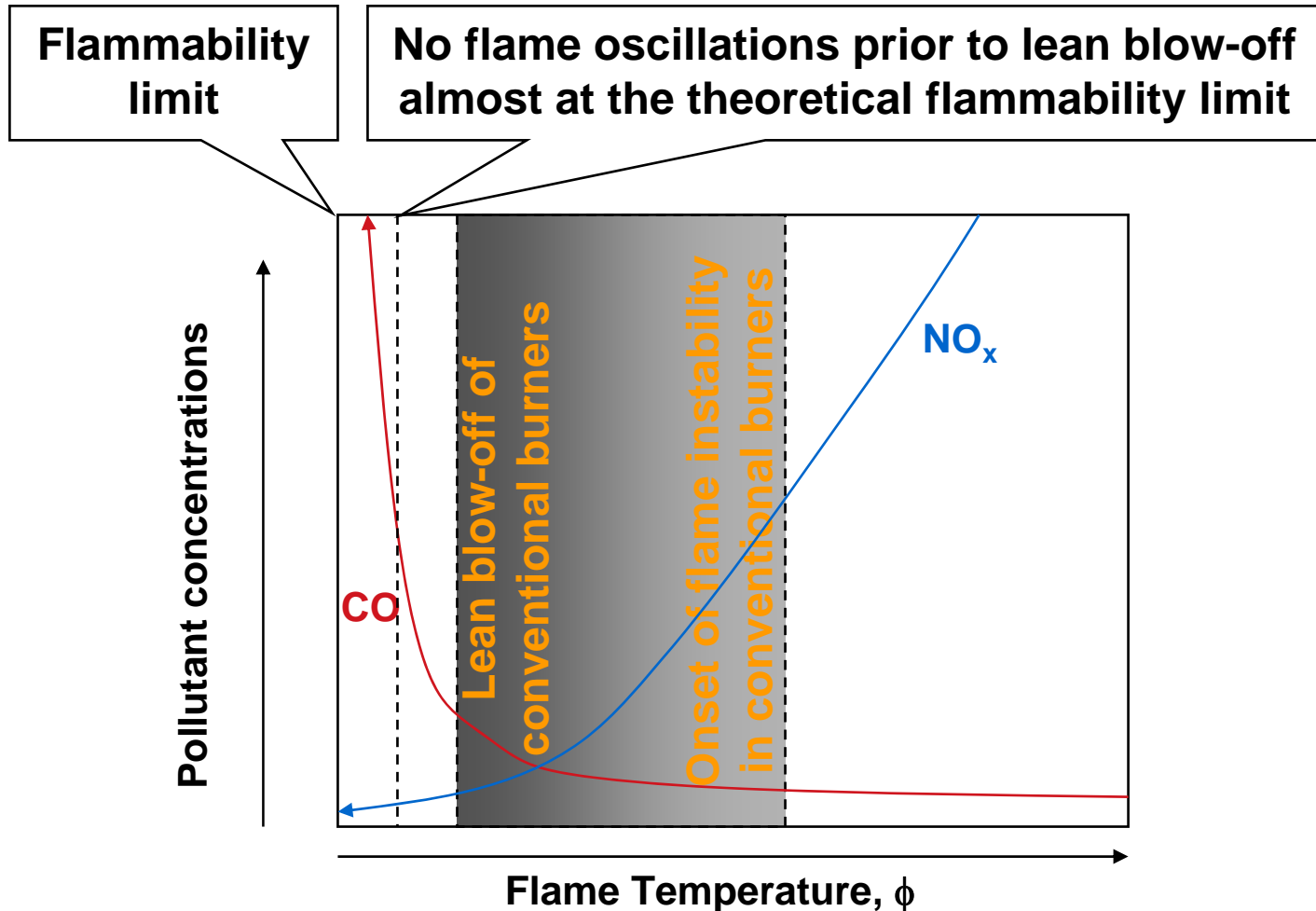
- Based on the theoretical principle of continuous ignition source provided by a **pilot flame** or in the hot **recirculation zone** behind the stabilizer



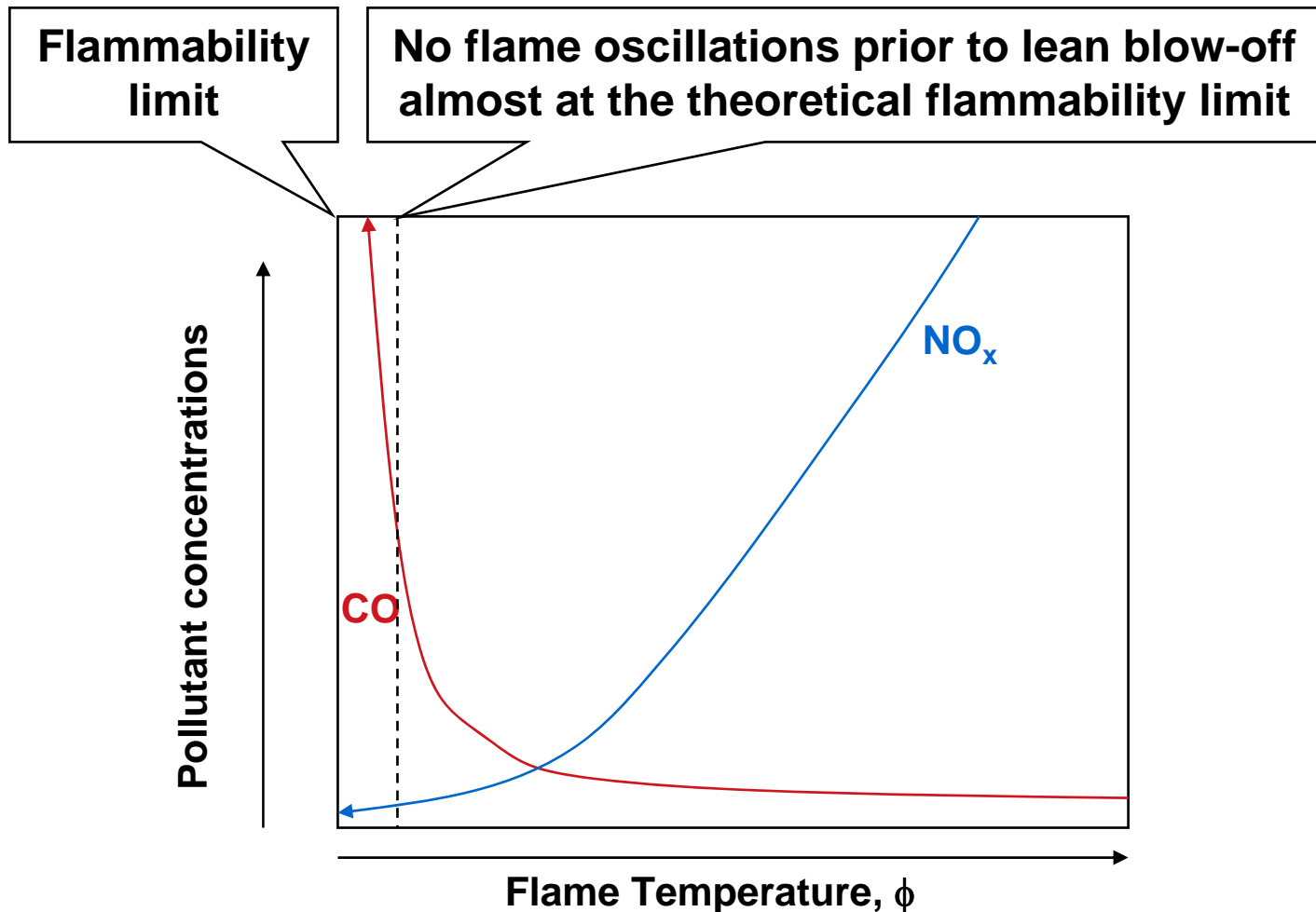
Lean Blow-off and Flame Instabilities Associated With Different Flame Holders Are Barriers to Reaching Low-Emissions



Low-swirl Combustion Exploits Aerodynamics to Overcome the Barriers to Attaining Low-Emissions



Low-swirl Combustion Exploits Aerodynamics to Overcome the Barriers to Attaining Low-Emissions



Premixed Flames Stabilized by Low Swirl

- ***Novel concept discovered in 1991 at LBNL***
 - ***Defies recirculation theory on flame stabilization***
- ***Scientific Interest***
 - Scientific background lacking for low-swirl flows
 - Challenging modeling problem
 - Excellent laboratory research tool
- ***Technological Interest***
 - Capability to support ultra-lean flames
 - Simple design
 - Patent awarded 1998

Flame Holders Were Considered Essential to Anchor Lean Premixed Flames

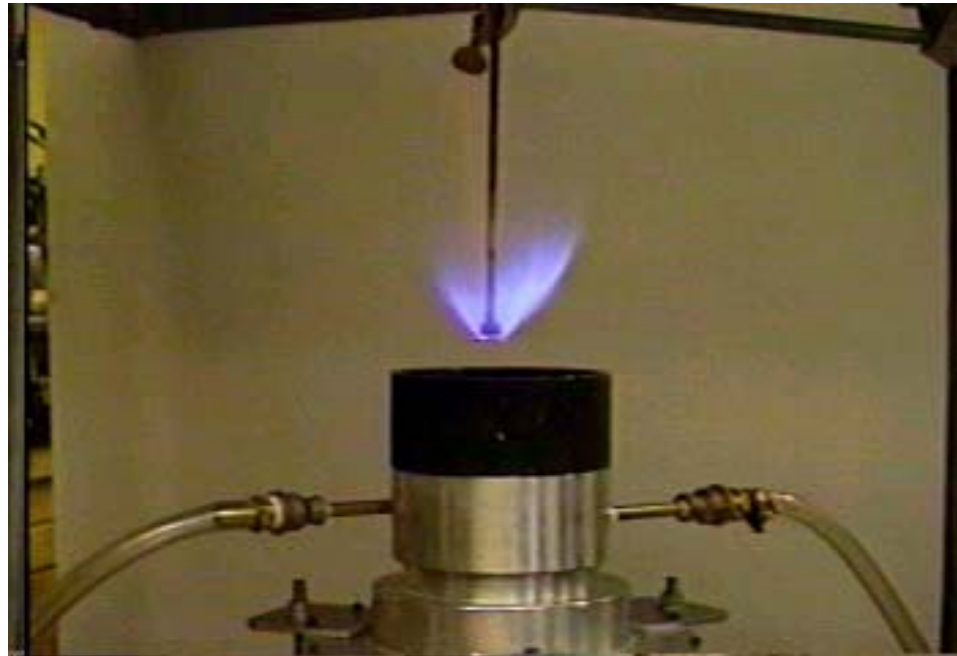
- Premixed flames requires a physical stabilizer so that it can anchor. This flame is stabilized by a bluff body of about 1 cm diameter



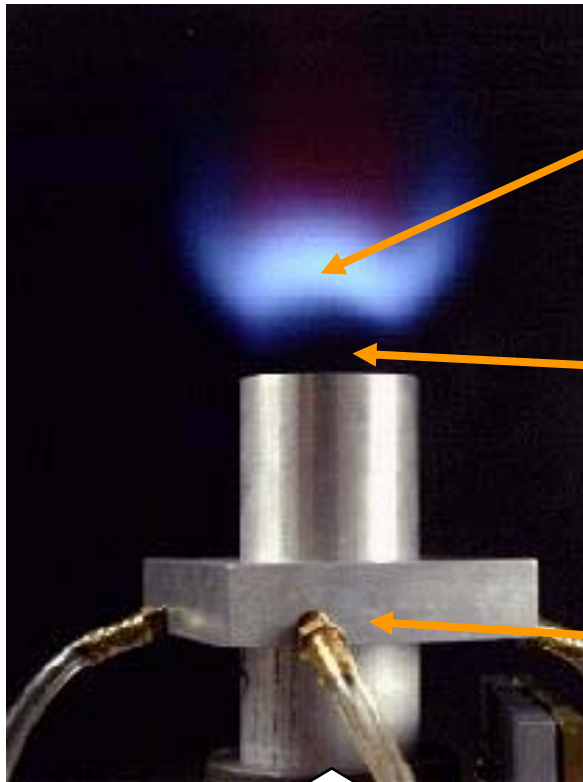
Reactants

Low-Swirl Eliminates the Need for a Flame Holder

- By introducing a very small amount of swirl air (swirl number $S \approx 0.6$), this video shows that the flame can self propagate without the bluff body



Low-Swirl Flame Stabilization Exploits Propagating Nature of Premixed Flames



Fuel/Air
mixture

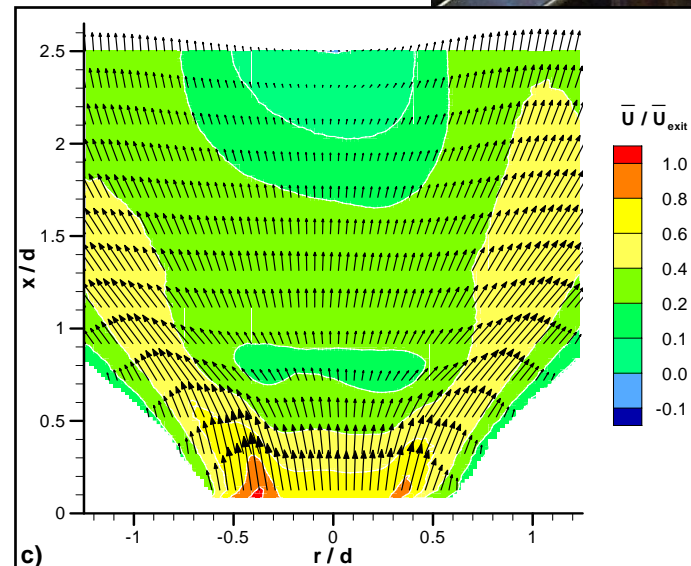
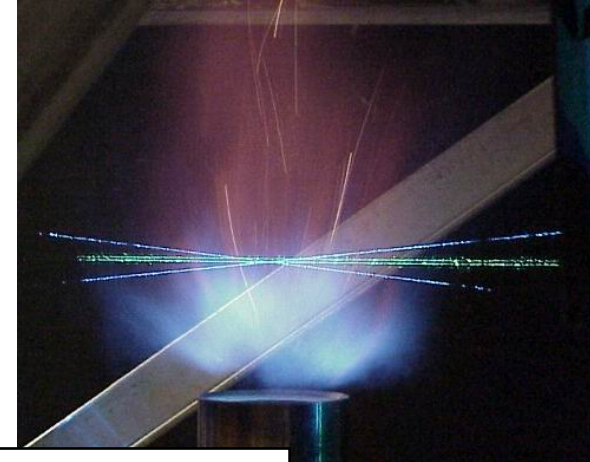
Propagating against the divergent flow, the flame settles where the local velocity equals the flame speed

Flow divergence (generated by low-swirl) above the burner tube is the key element for flame stabilization

Small air jets swirl the perimeter of the fuel/air mixture but leave the center core flow undisturbed

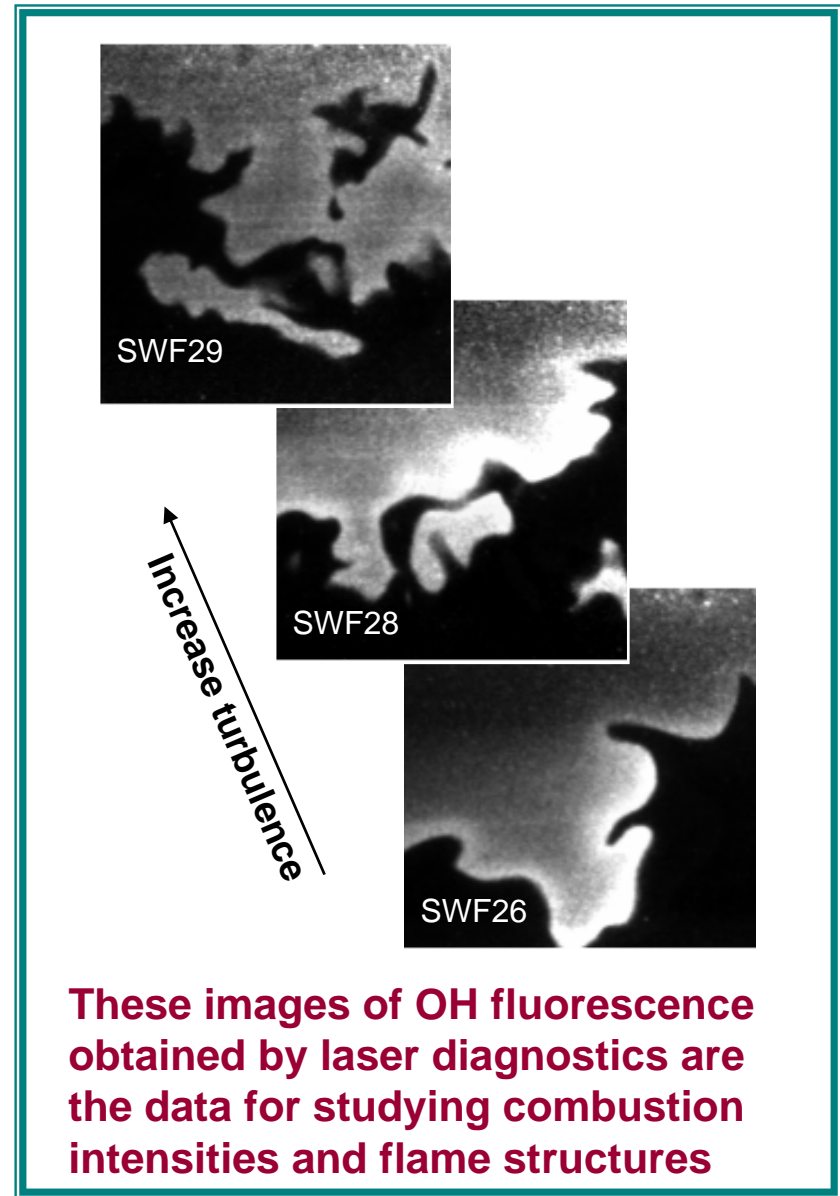
Laser Diagnostics Characterized Flame Stabilization Mechanism

- Flow divergence provides a much more stable mechanism for lean flames than high swirl flows or flame holders
- Flame brush propagates at turbulent flame speed that increases linearly with turbulence intensity
 - flashback conditions predictable
- Swirl intensity controls flame lift off position



Scientific Studies Using Jet-LSBs

- Exploiting LSB's capability to supports premixed turbulent flames under a wide range of turbulence and mixture conditions has helped to resolve key scientific problems
 - Investigate evolving turbulent flame structures from low to intense turbulence
 - Verify new theory on classification of premixed turbulent flames
 - Relate turbulent flame speed to combustion intensity

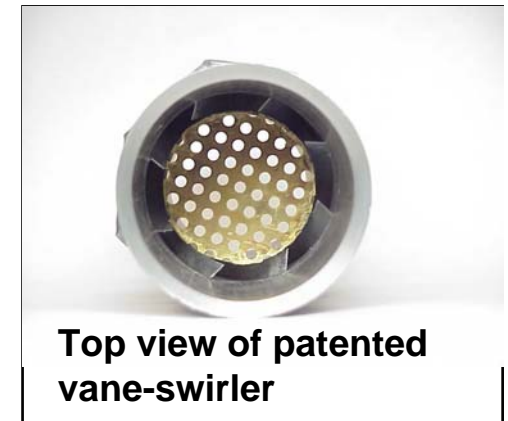
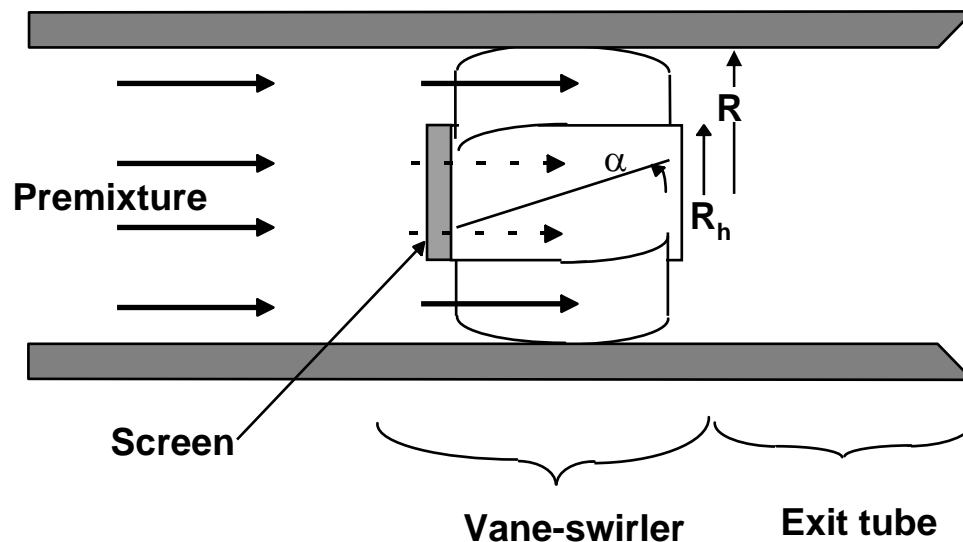


First Technology Transfer Project Supported by DOE-LTR (1994-1997)



- ***CRADA with Teledyne-Laars***
- ***Adaptation to pool heaters***
 - Design and test LSBs that meets the operational requirements of 15 KW to 100 KW units
 - Sizes similar to laboratory LSBs
 - Non-modulated systems, no turndown requirement
- ***Issues***
 - ***need simpler design requiring only one flow supply (no jets)***
 - firing sideways or downwards to attain > 85% efficient
 - stable inside chamber
 - cannot compromise on energy efficiency
 - cost must be lower than Alzeta burner (\$100/per unit)

Vane-Swirler Developed for LSB

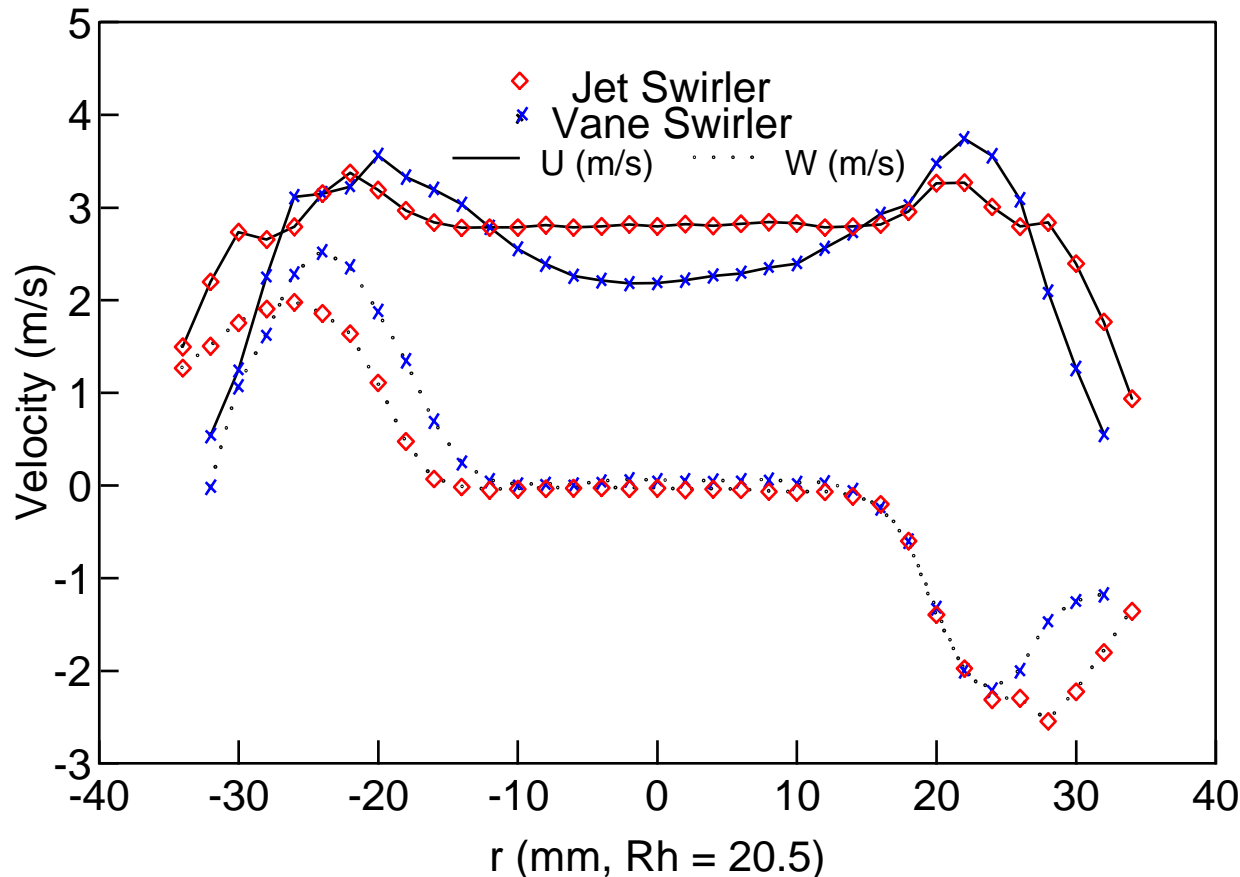


□ *New design fundamentally different than that of conventional vane-swirler*

- Open center channel allows a portion of flow to bypass swirl vanes
- Angled guide vanes induce swirling motion in annulus
- Screen balances pressure drops between swirl and center channel

□ *Patent awarded in 1999*

Development of Vane-Swirler Relied on Laser Diagnostics



- Varied screens blockage, vane angle, and inner tube diameter
- Measured mean velocity profiles and compare with profiles of jet-swirler

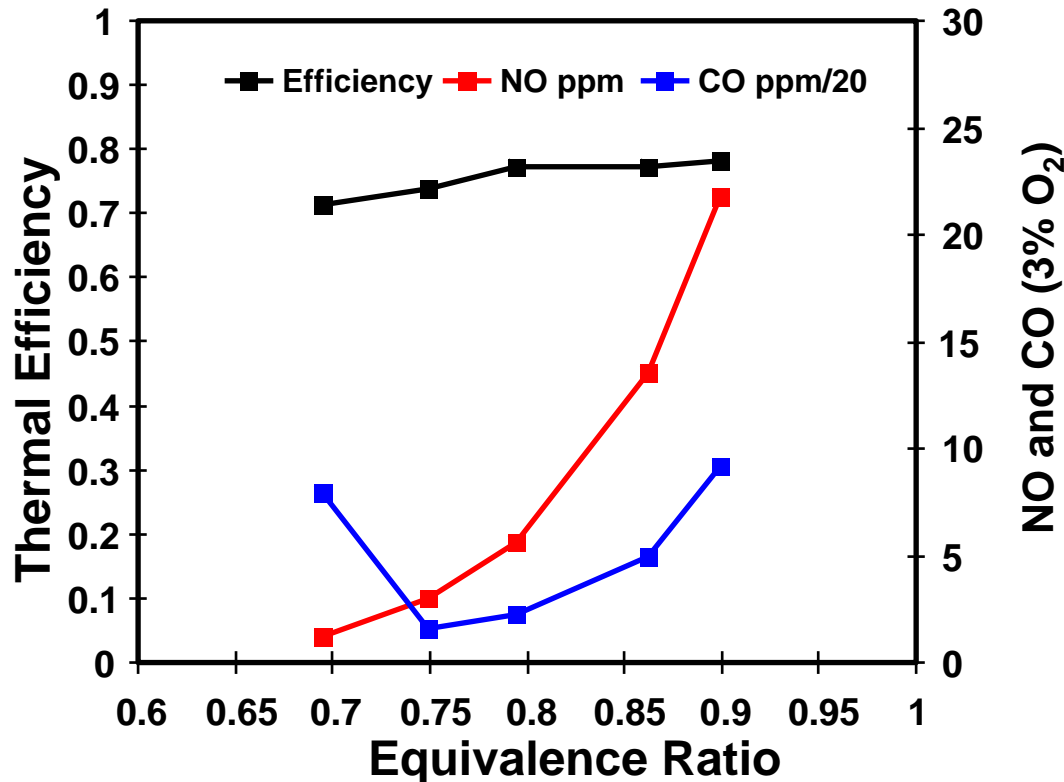
Vane-Swirler for LSB Can Be Made From Simple and Low-Cost Materials



This burner is made of PVC and plastic to showcase the uniqueness of LSB

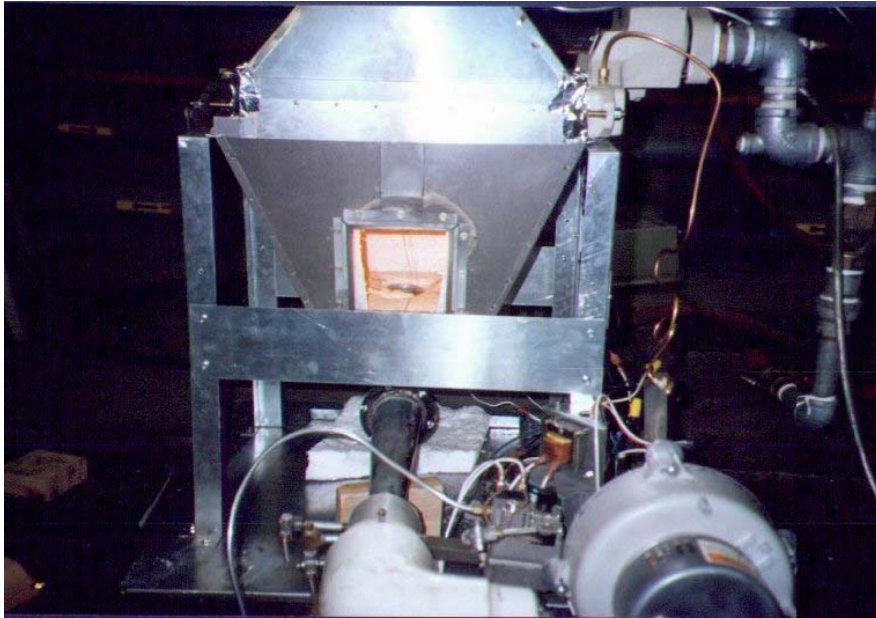
- ☐ Vane-LSB produces the same shape as Jet-LSB
- ☐ Lifted flame does not transfer heat to burner throat
- ☐ Estimated fabrication cost for pool heaters < \$10/unit

Pool Heater with Low-Swirl Burner Achieved < 10 ppm NO_x



- Flame remains stable inside chamber
- No compromise on thermal efficiency
- CO can be further reduced
- LSB compatible with ignition and control systems of current products

Technical Success But No LSB Product!



**Prototype with a 75 kW LSB
(250,000 BTU/hr)**

- ❑ Clean and highly efficient pool heaters represent a small sector of T-L product line
- ❑ Growing market in Southwest regions (e.g. AZ) has no emission regulations
- ❑ Market needs in California can be met by out-sourcing to burner suppliers (Alzeta)
- ❑ R&D dollars re-directed to updating product appearance (signature look) and improving ease and convenience of operation (remote control)

Scaling to Industrial Sizes (CIEE 1997-2001, DOE-OIT 2000-present)

□ ***Established adaptability to process heat and boilers***

- Targeting single burner ranging from 0.5 MW to 20 MW
- Starting at a minimum of 6X scale up from LSB for pool heaters

□ ***Obtained scaling information***

- Lacking scientific background information for low-swirl flows
- Theory on turbulent flame speed predicts flame blowout
- Possible trade-off and/or compromise between two scaling approaches
 - ♦ increasing flow velocity versus increasing burner diameter

Learn From Equipment Manufacturers

- ***Presentations at American Flame Research Committee meetings***
- ***Discussion with R&D engineers and managers***
 - Site visits and demonstrations
- ***Outreach and publicity***
 - LBNL Technology Transfer booth at trade shows
 - Appearance in “Your New Home” on Discovery channel

Key Scaling Questions

- ***What are the critical design components of the LSB?***
 - Size of center channel?
 - Exit tube length?
 - Vane angle?
 - Vane length?
 - Screen placement position?
 - Homogeneity of mixture?
- ***How high we can push the throughput?***
 - Do we need to adjust swirl to accommodate flame shift?
 - Will the flame blows out as in other burners?
 - How does the aerodynamic flowfield evolve at high velocities?
- ***How much can we increase the burner diameter?***
 - Will increase burner diameter affect flame stability range and thus swirl requirement?
- ***Is there a convenient scaling rule that engineers can use?***

Critical First Step – Quantify Swirl Rates By New Swirl Numbers Derivations

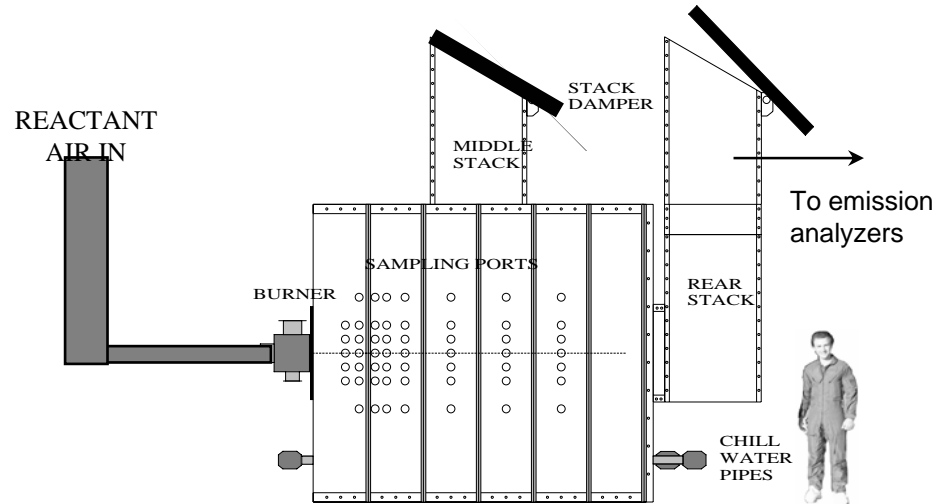
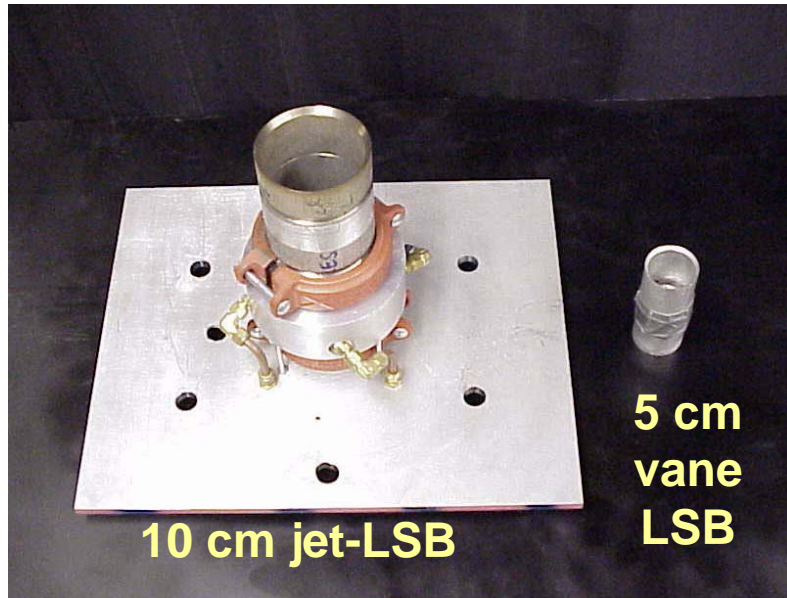
- For air-jet swirler S defined in terms of flow rates

$$S = \frac{\pi R^2}{4\pi R_j^2} \frac{\dot{m}_j^2 \cos \alpha}{(\dot{m}_i + \dot{m}_j)^2}$$

- For vane-swirler, momentum integrations gives an equation in terms of flow velocities
 - not a convenient form for engineering application

$$S_v = \frac{2}{3} \tan \alpha \frac{1 - (R_c / R)^3}{1 + (R_c / R)^2 \left((U_c / U_a)^2 - 1 \right)}$$

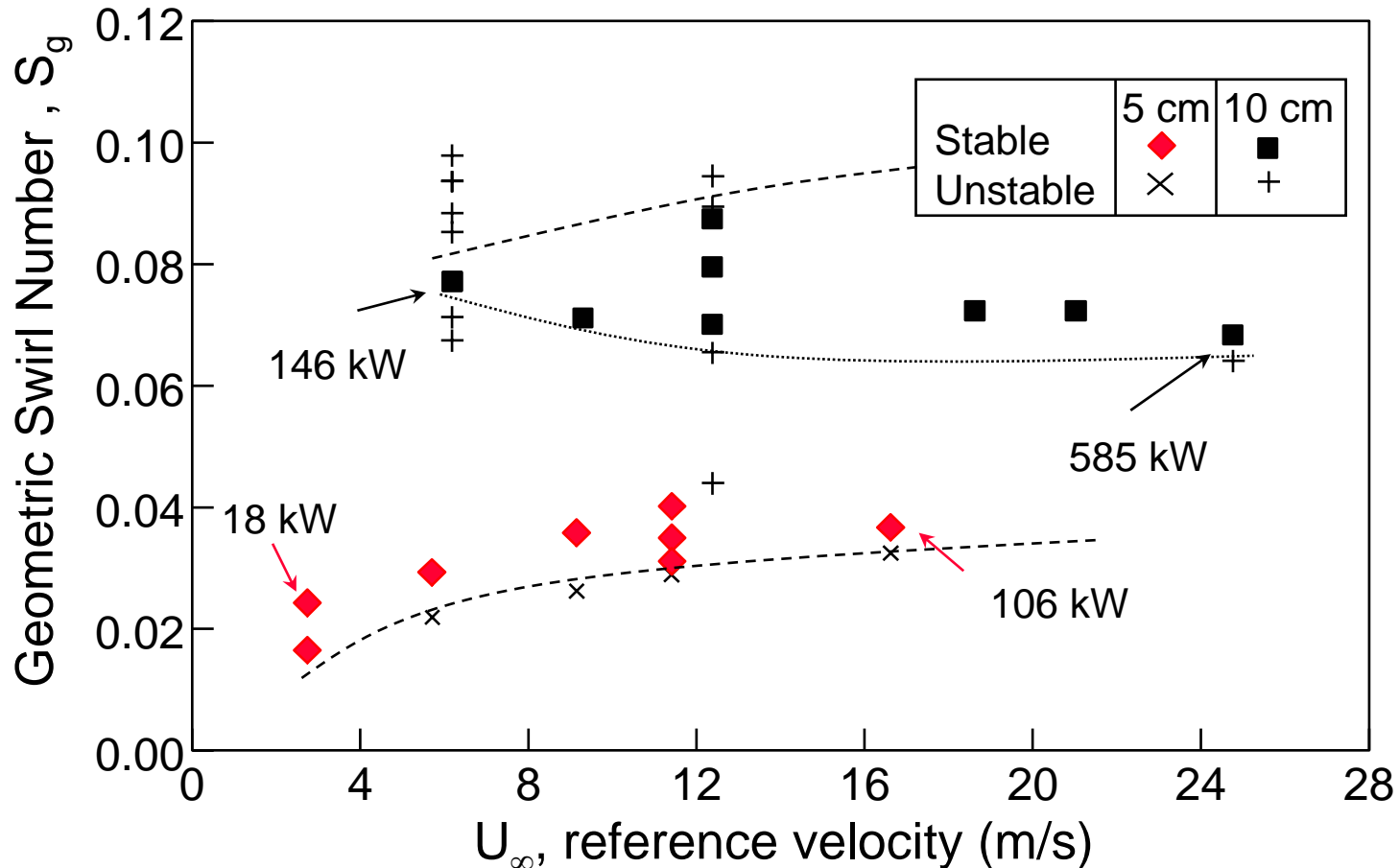
Obtaining Scaling Information Through Laboratory Studies



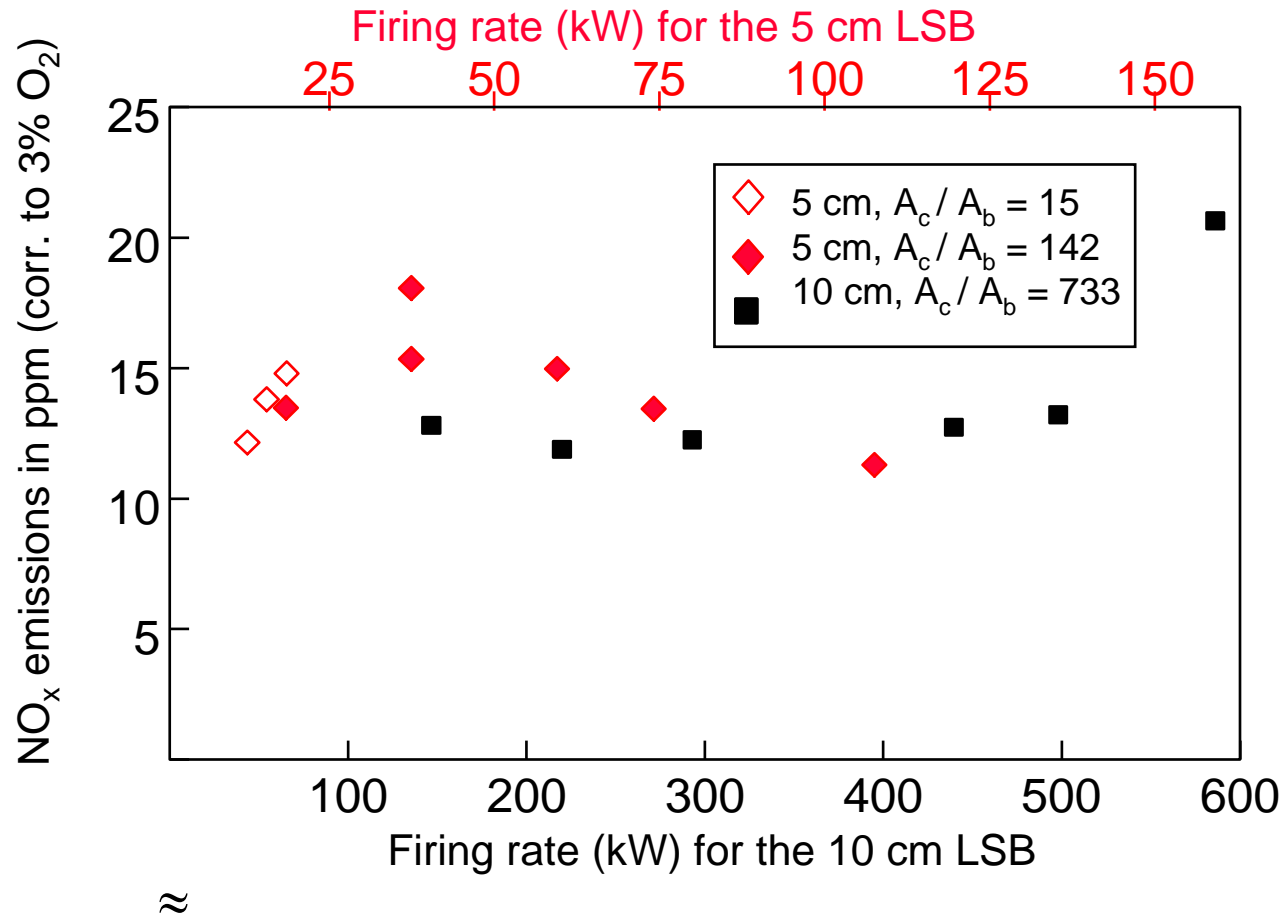
- Comparing LSBs of different sizes
- 10 cm jet-LSB has dimensions twice those of 5 cm jet-LSB
 - Tested at furnace simulator 150 to 600 kW ($6 < U < 25$ m/s)
 - Determine swirl number, lean blow off and emissions
 - Compare results with 5 cm jet-LSB

Swirl Requirement Independent of Load

- Vane-swirler does not require adjustments for load change
- Swirl rate subscribes to residence time scaling

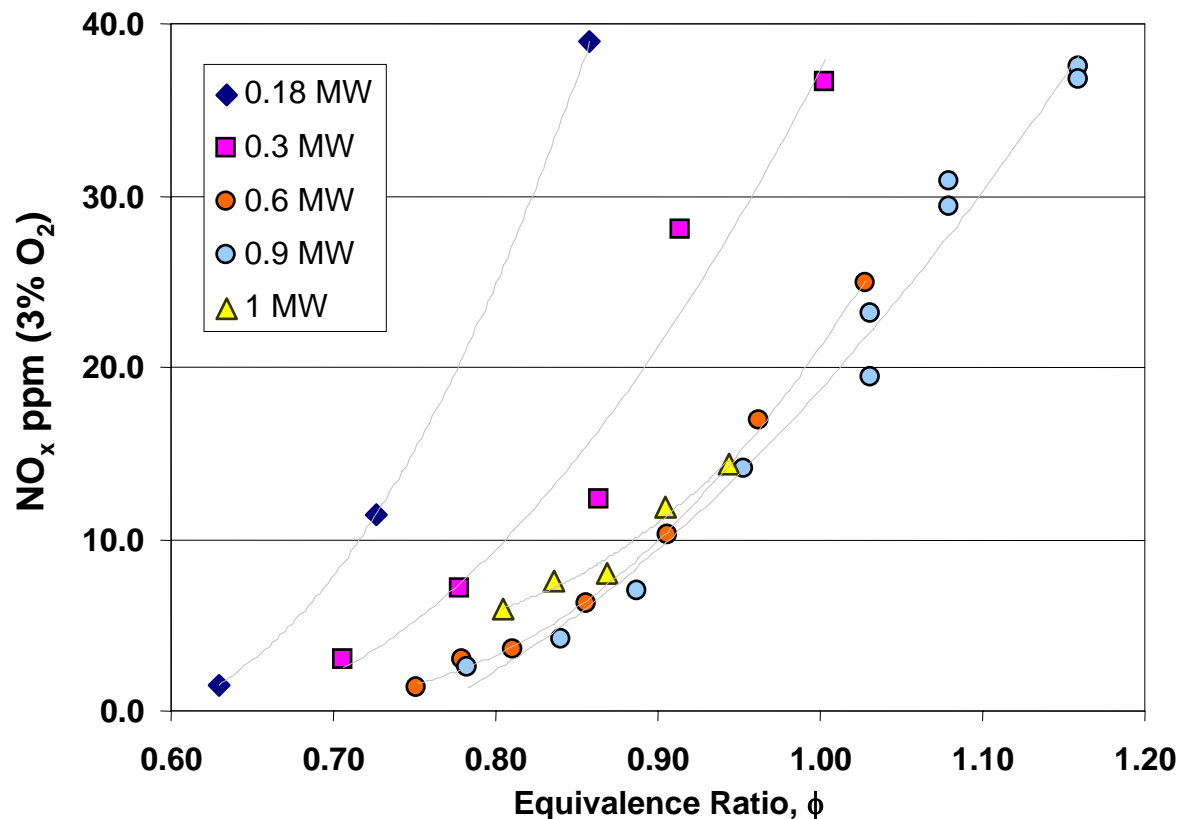


NO_x Emissions Independent of Burner Size and Velocity

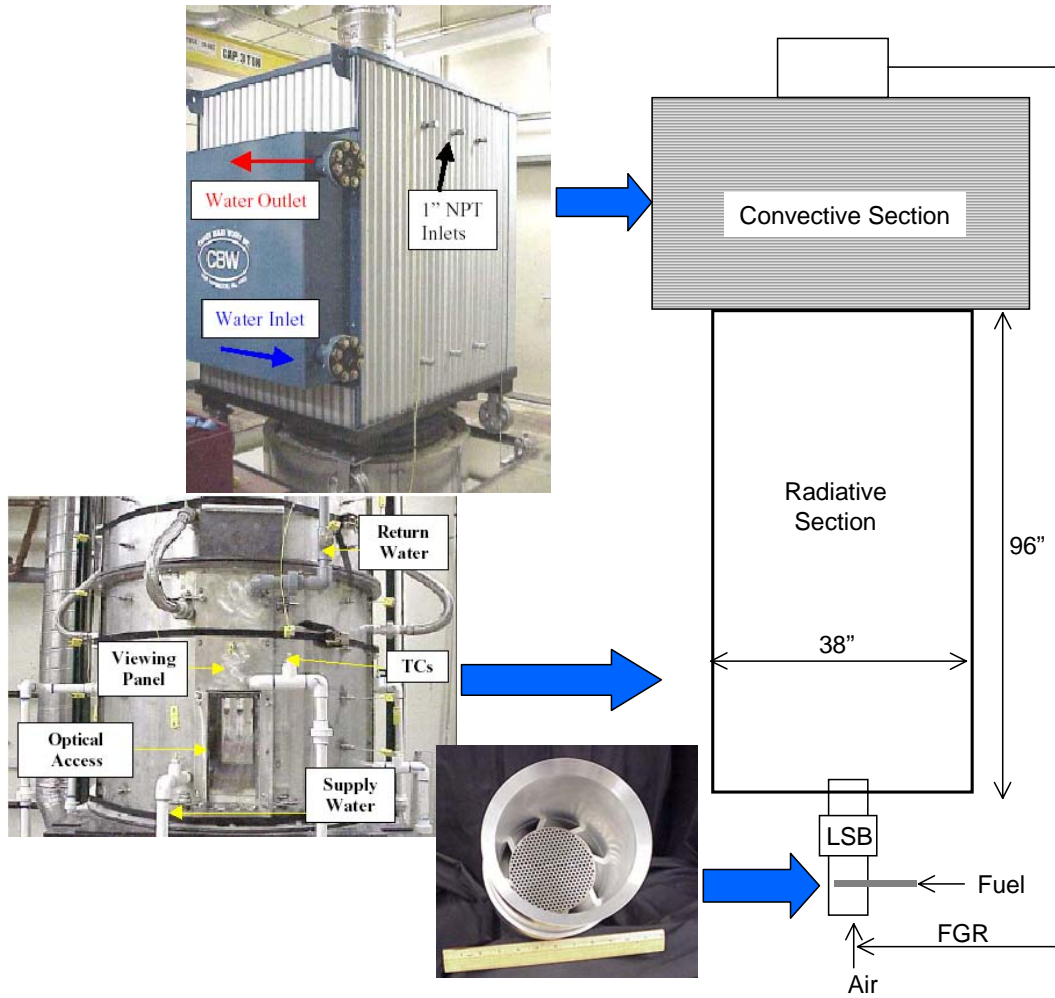


Continue Development of Vane-Swirler

- Larger 7.7 cm LSB reached 1 MW in furnace simulator
- $\text{NO}_x < 10 \text{ ppm}$, $\text{CO} < 25 \text{ ppm}$ and UHC undetectable



Further Scale-up to 12.7 cm and Investigated Effects of Vane-design



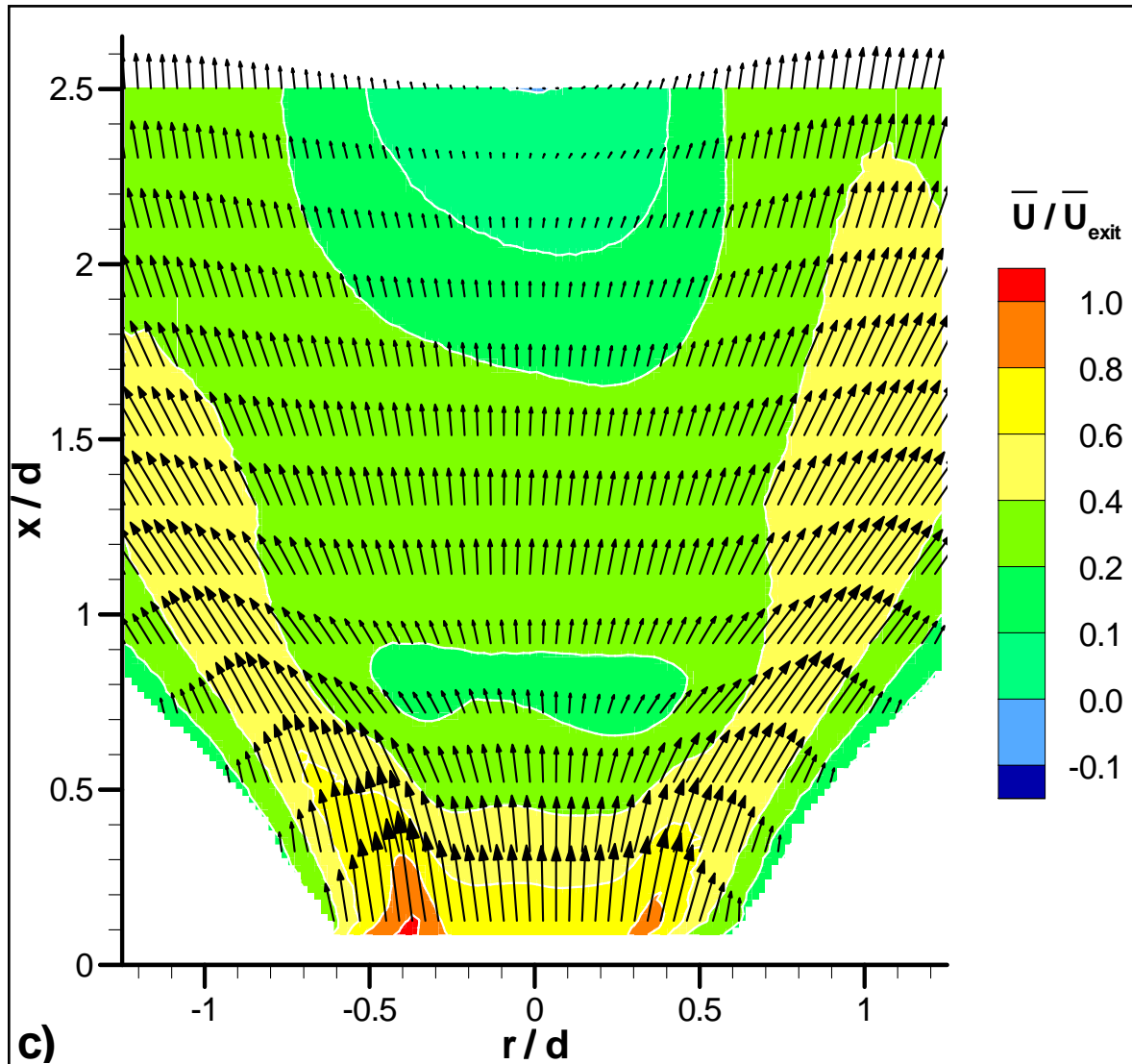
12.7 cm LSB fitted in boiler simulator at UC Irvine

- 12.7 cm LSB with four vane types
 - curved vanes $\alpha = 45^\circ$
 - curved vanes $\alpha = 37.5^\circ$
 - straight vanes $\alpha = 37.5^\circ$ short
 - straight vanes $\alpha = 37.5^\circ$ long
- For $0.7 < \phi < 0.9$, 0.6 to 1.3 MW emissions and burner performance independence of vane design & number of vanes

Laser Studies Provided Important Scientific Clues for LSB's Robust Performance

- Analyses drawn upon the theories on
 - Turbulence scaling, production, and dissipation
 - Flame temperature, flame speed and reaction chemistry
 - Combustion aerodynamics
- Found LSB generates self-similar flowfield
 - Flow divergence constant in non-dimensional space
 - No flame shift due to linear scaling of turbulence intensity and flame speed
- Knowledge essential for identifying, prioritizing and resolving operational issues
 - Placement of flame ignitor
 - Protocol to maintain flame stability during turndown and turnup
 - Premixing requirement
 - Flow conditioning upstream

Velocity Vectors of LSB Show How It Prevents Flash-back

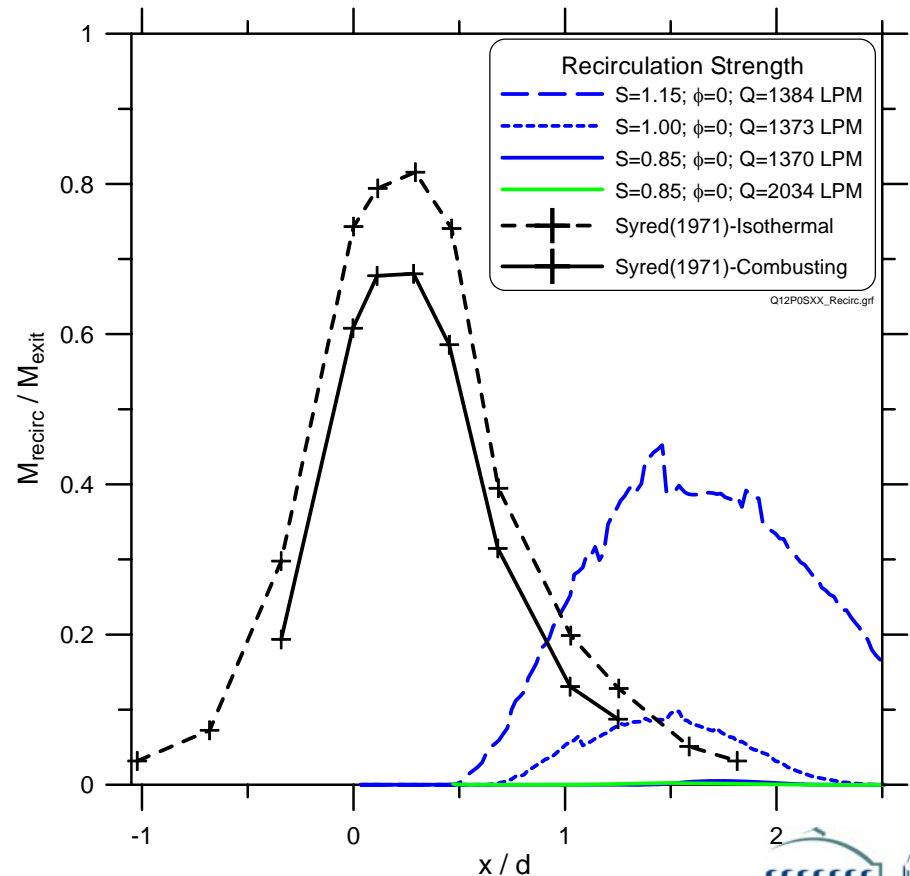
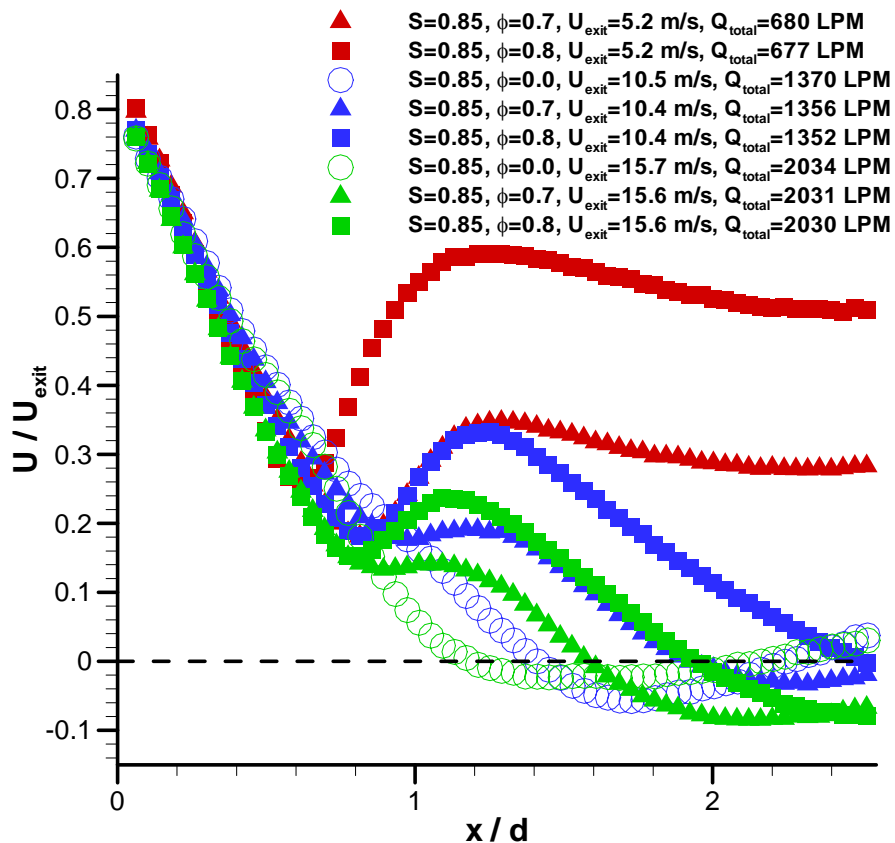


□ Particle Image Velocimetry (PIV) measurements show relatively uniform high inflow velocities with no back-flow



Found Self-Similarity Flowfield and Weak Downstream Recirculation

- Self-similarity explains why flame does not shift with load
- Weak recirculation shows it to be irrelevant



Refinement of Swirl Number Definition for Combustion Engineers



$$S = \frac{2}{3} \tan \alpha \frac{1 - R^3}{1 - R^2 + [m^2 (1/R^2 - 1)^2] R^2}$$



□ New expression uses easily measurable parameters

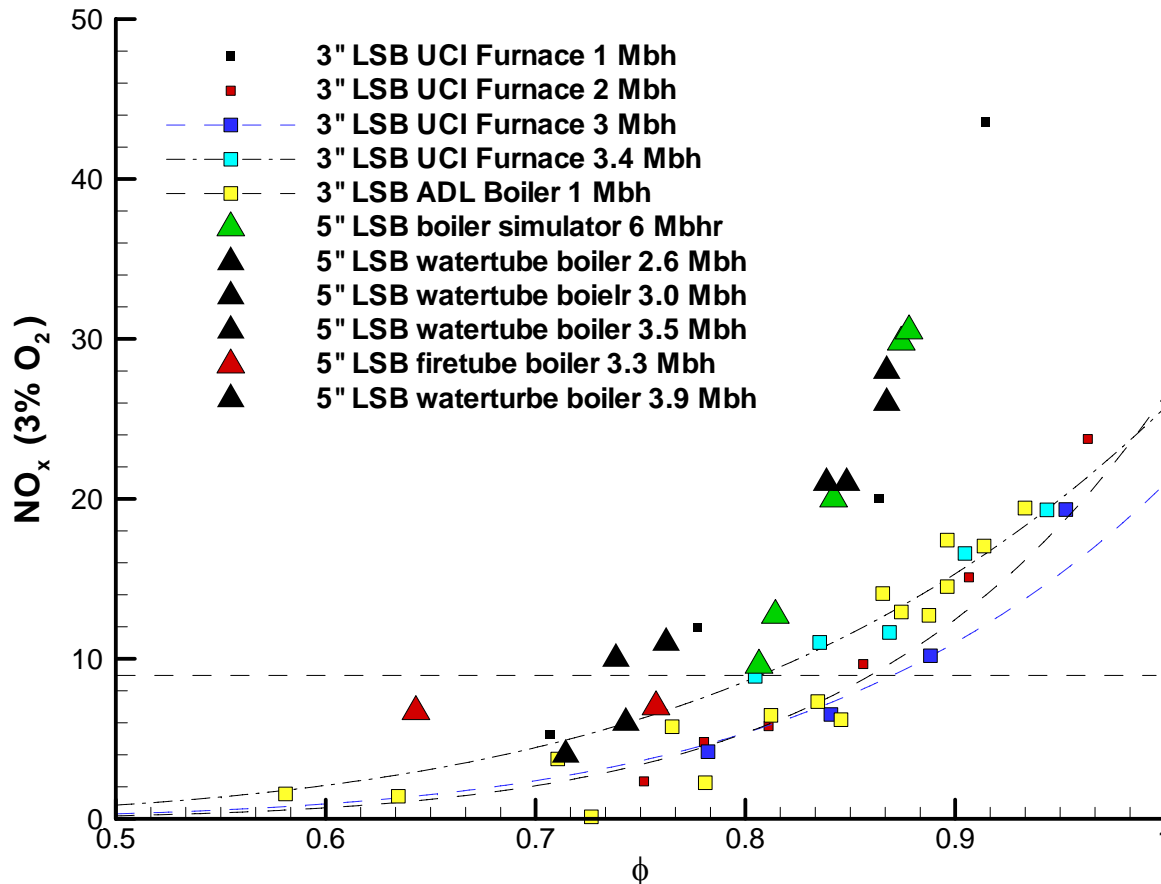
- Ratio of center channel radius to burner radius, $R = R_c/R_b$
- Straight or curved vane with angles, α
- Ratio of mass flow rates through center channel and swirl annulus, m
 - Standard pressure drop procedure to obtain m from different screens

Developed Engineering Guidelines

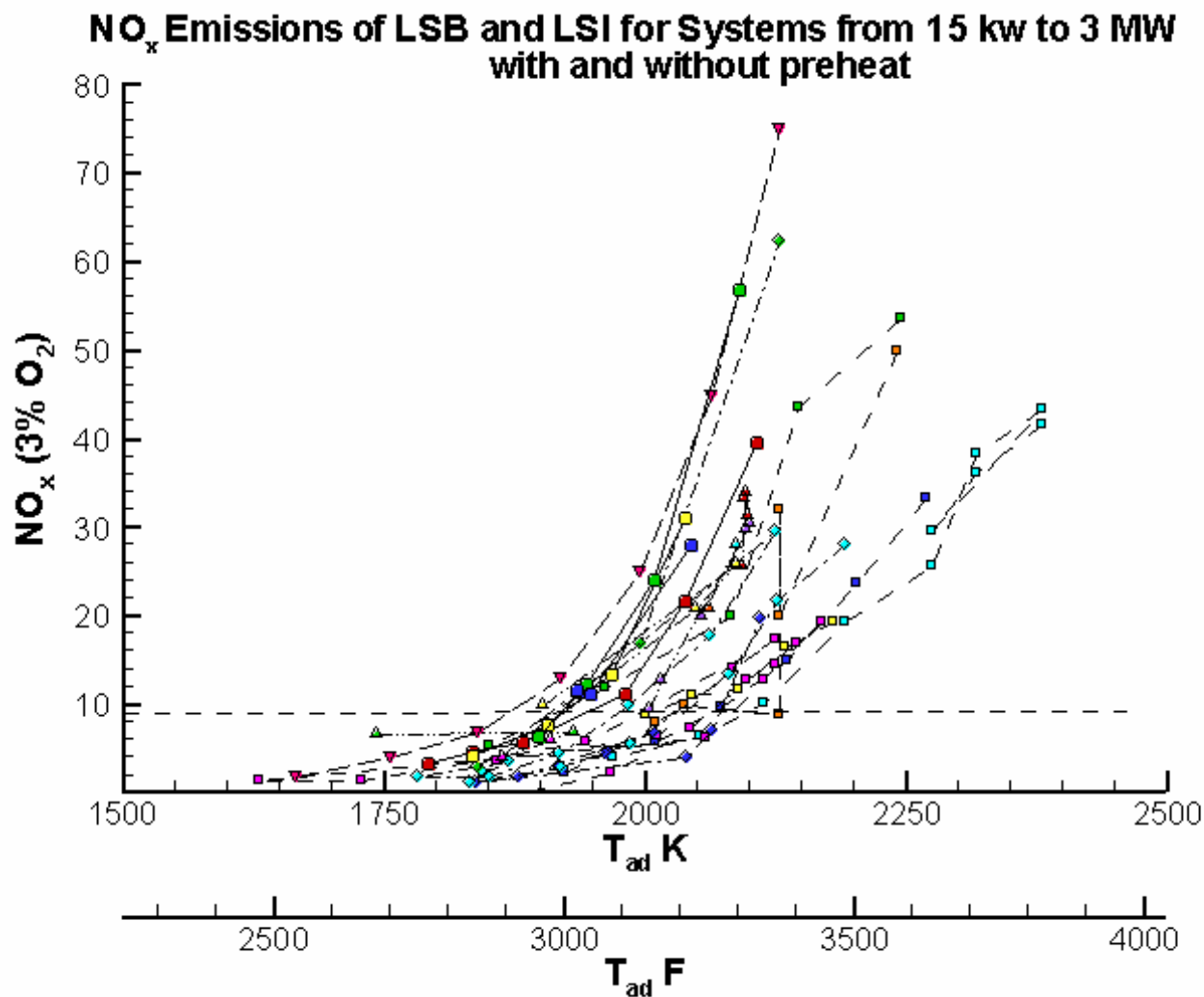
- Keep swirler recess at 1 to 1.5 diameter
- Apply $0.4 < S < 0.55$ criterion
 - Tune swirler by using different screens to change S
 - Screen geometry is not important
 - ♦ Use larger openings to reduce clogging problem
 - ♦ Can explore other options to balance core and annulus flow
 - Vane angle between 37° to 45°
 - Vane can be curved or straight
- Constant velocity scaling applies
 - Minimum operating conditions predicted by the flash back point
- No need for elaborate and precise premixer

0.4 < S < 0.55 Criterion Scaled LSB to 40 cm and 13.5 MW

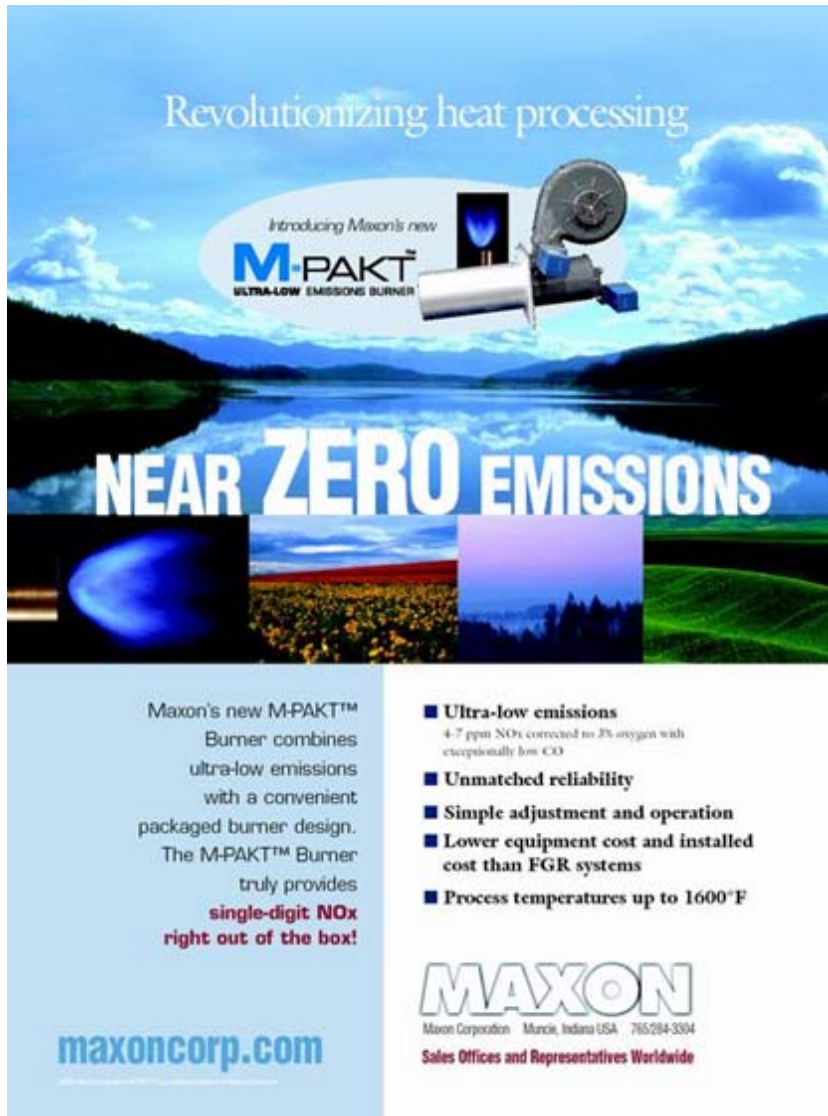
□ Consistent $\text{NO}_x < 9 \text{ ppm (3\% O}_2\text{)}$ in Industrial Systems



NO_x Dependence on Flame Temperature Shows Load Following Capability



Commercialization for Process Heat



Revolutionizing heat processing

Introducing Maxon's new
M-PAKT™
ULTRA-LOW EMISSIONS BURNER

NEAR ZERO EMISSIONS

Maxon's new M-PAKT™ Burner combines ultra-low emissions with a convenient packaged burner design. The M-PAKT™ Burner truly provides **single-digit NO_x** right out of the box!

- **Ultra-low emissions**
4-7 ppm NO_x corrected to 3% oxygen with exceptionally low CO
- **Unmatched reliability**
- **Simple adjustment and operation**
- **Lower equipment cost and installed cost than FGR systems**
- **Process temperatures up to 1600°F**

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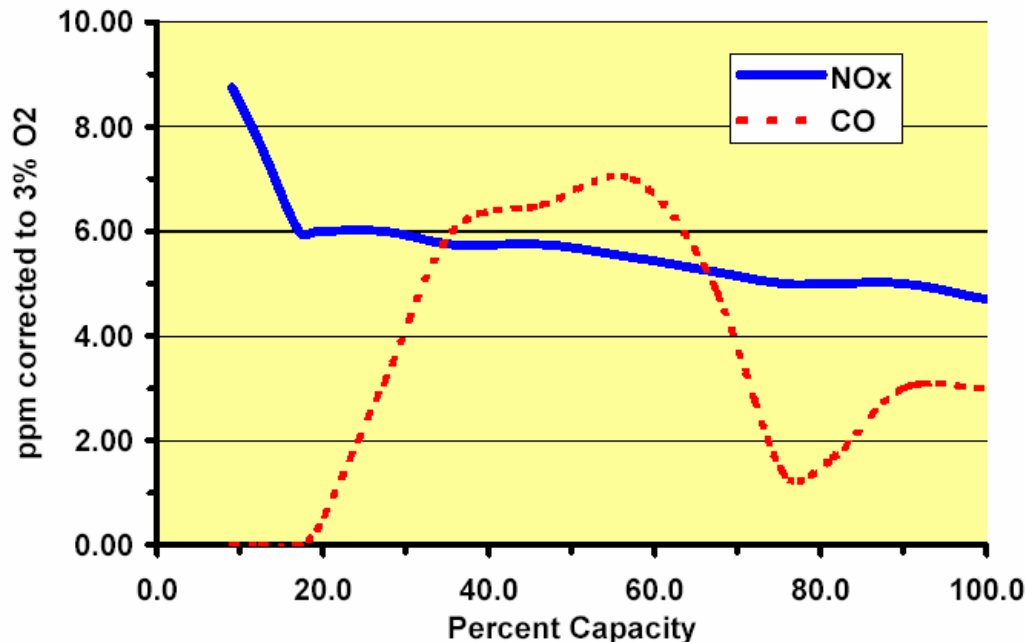
- Maxon Corporation licensed LSB in 2002 after a successful joint development and testing program with LBNL
- Target ultra-low NO_x market (< 9 ppm at 3% O₂ guaranteed) for industrial heating, baking and drying
- 300 kW to 1.8 MW (1 – 6 MMBtu/hr) products in production
- Tested LSB at 45 MMBtu/hr

M-PAKT Burner Maintains Ultra-Low Emission Throughout Load Range

M-PAKT™ Ultra Low NOx Burner

- Simple and compact premixer
- Conventional controls

Typical Emissions



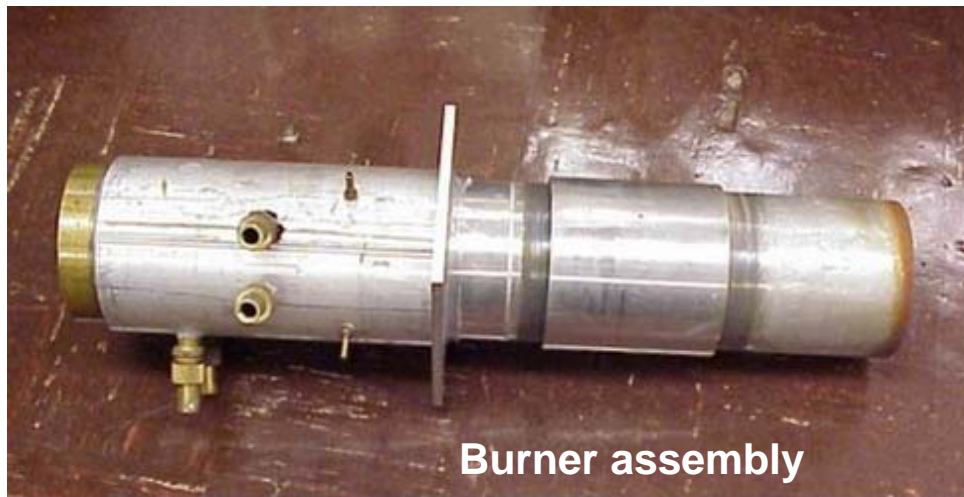
Maxon Identified Significant Economic and Technical Advantages of LSB

- ***Design scales by governing equations***
 - A radical departure from experimentation approach
- ***Size compatible to existing equipment***
- ***Fabricated with no initial re-tooling or new patterns***
 - Fewer parts from common materials
- ***Use existing controls for conventional burners***
- ***Flame not in contact with burner tip***
 - No thermal stresses to burner that causes metal fatigue
- ***Lower operational cost, and greater ease of operation, thanks to simpler combustion process***

Addressing Energy Efficiency Issues and Further Lowering Emissions

- ***Flue gas recirculation (demonstrated)***
 - Tested in boiler with external FGR
 - TIAx developed and tested internal FGR/premixer
- ***Reduce power of fan blower (accomplished)***
 - At least 50% reduction with no change in LSB performance
- ***Partial reforming to reach < 2 ppm NO_x (demonstrated)***
 - Traces of H₂ enhance flame stability and lower CO
 - Steam reformer or MIT's plasmatron
- ***Highly preheated combustion (demonstrated)***
 - Waste heat recovery
- ***Staging and burner/chamber coupling (planned)***

5" (12.7 cm) LSB for Boiler Testing



Burner assembly

- Two 5" LSB prototypes
 - 0.5 – 7 MMBtu/hr capacity
 - $R = 0.8$ with 8 vanes
 - $R = 0.6$ with 6 vanes
- **10" LSB for up to 30 MMBtu/hr available in 02/2004**



Gas injectors



Flame lift-off height adjusted for boiler enclosure

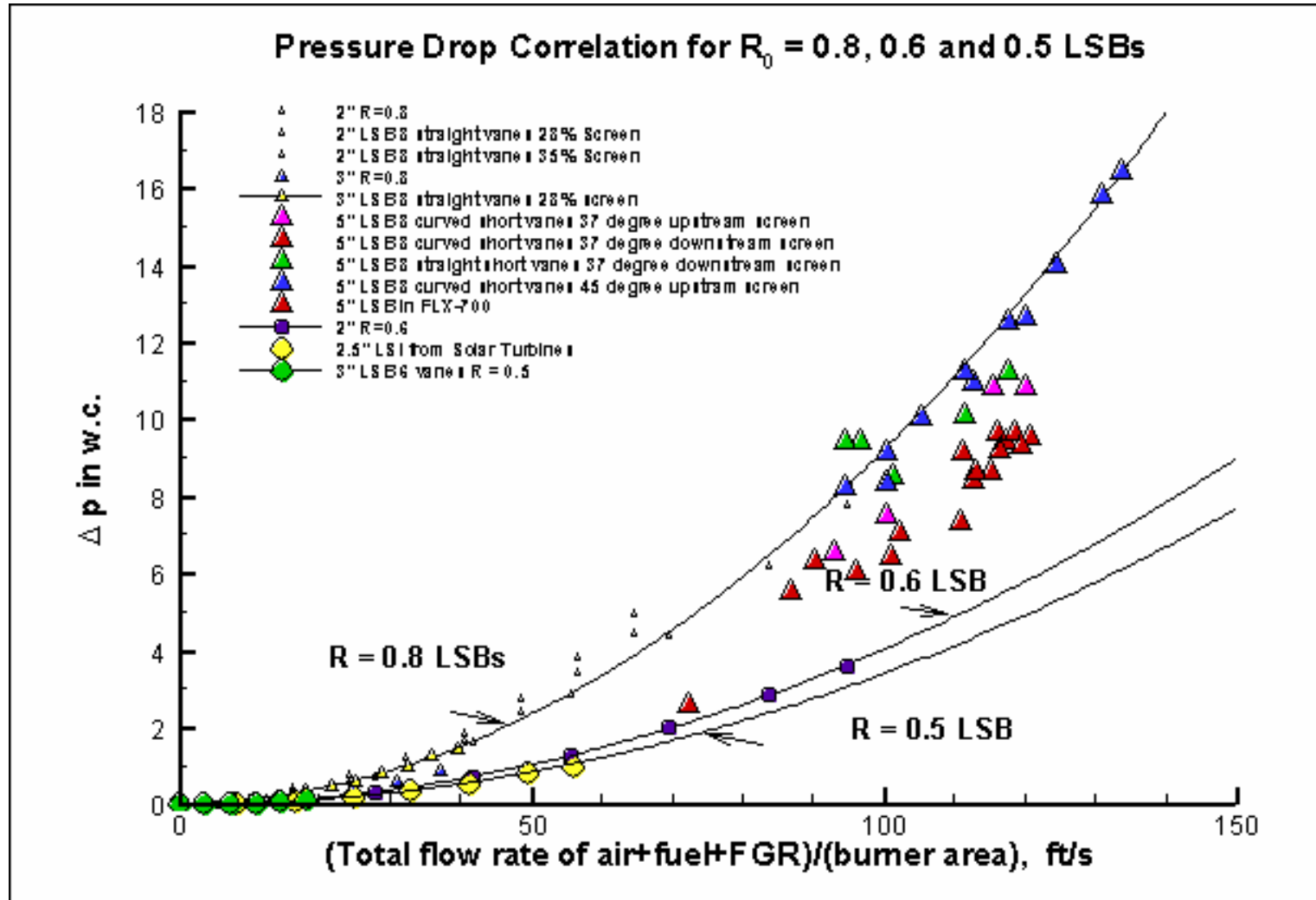
Evaluate 12.7 cm LSB in Commercial Watertube Boiler with External FGR



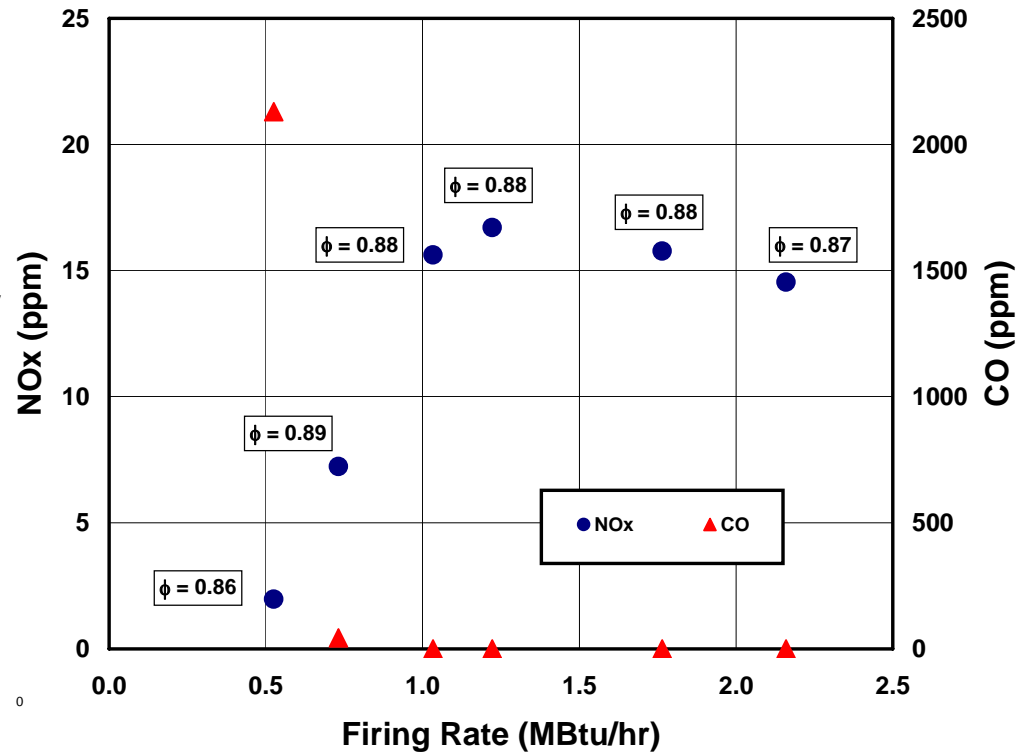
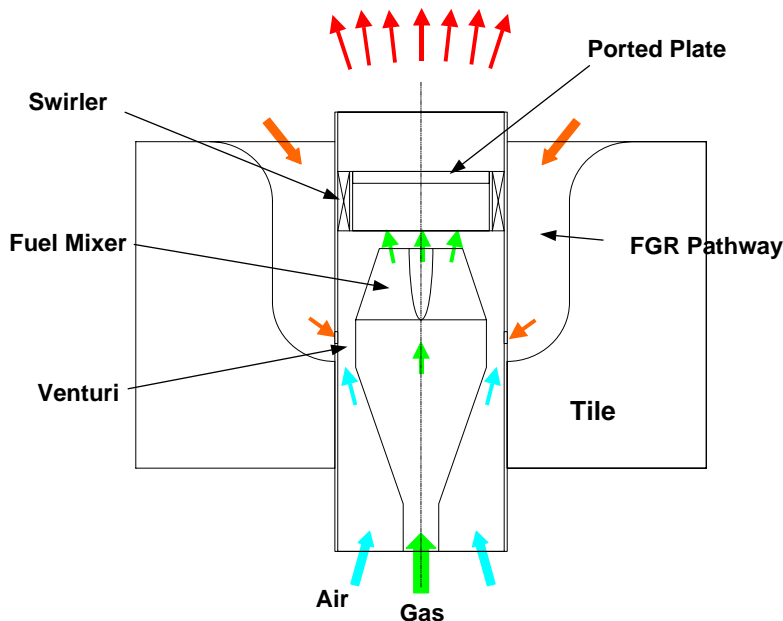
- Use blower and controls for the Cleaver-Brooks Boiler
- Performance targets
 - $\text{CO} < 12 \text{ ppm}$ and $\text{NO}_x < 9 \text{ ppm}$
 - .15 to 2 MW, 5:1 turn-down
 - $\phi > 0.87$, $< 35\% \text{ FGR}$
- LSB exceeded most targets
 - $< 9 \text{ ppm NO}_x$ with $< 12\% \text{ FGR}$ at $\phi = 0.87$ at 1.2 MW
- Fan pressure requirement slightly high

Smaller R Reduces Back Pressure

- Obtained equation for sizing fan requirement



ADLittle Developed and Tested Venturi Premixer and FGR Entrainer for LSB



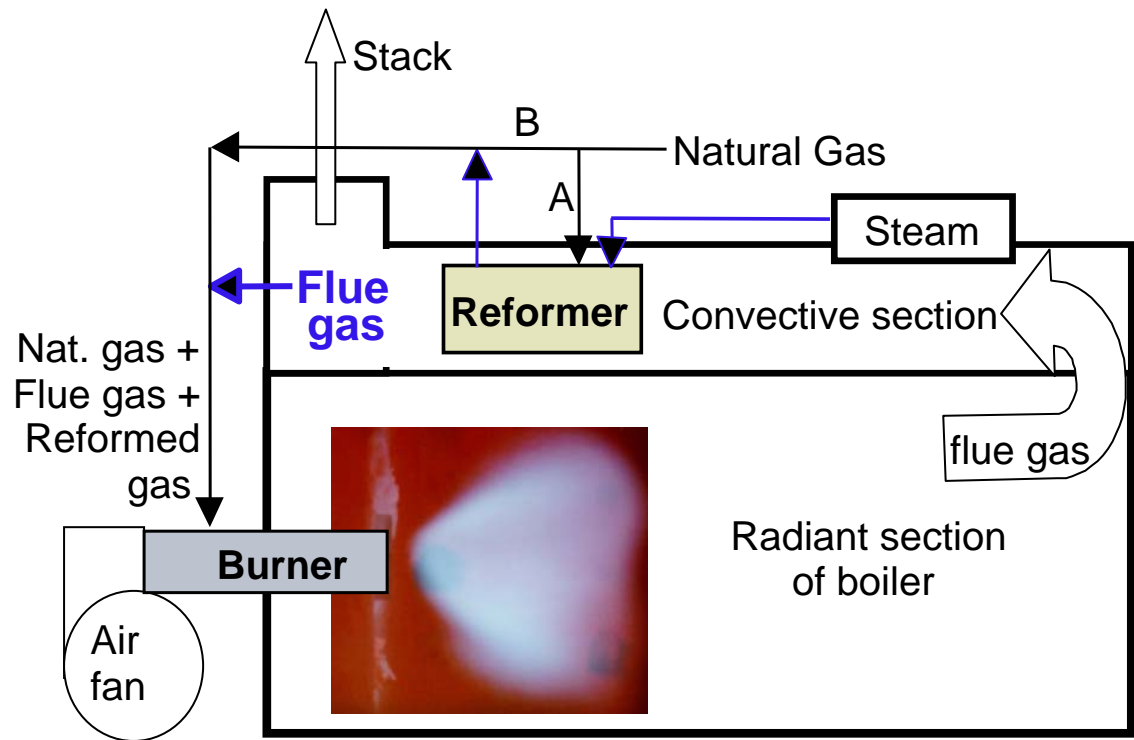
- Computation fluid dynamics (CFD) to optimize design
- Applied scaling equation to size LSB at 19 cm

< 5 ppm NO_x Concept -- FGR + LSB + Partially Reformed Natural Gas

- Exploit combustion of hydrogen enriched natural gas
 - Use LSB to capture these benefits
 - Partial reformer to produce optimum H₂:CH₄ ratio in fuel

- Demonstrated in 15 kW water heater simulator

- $0.7 < \phi < 0.9$
 $0 < \text{FGR} < 0.3$
PRNG = 0 and 0.05
- Reformer at 650 C
CH₄ = 0.12 l/s
steam = 0.04 l/s
- Steam (~5%) has no effect on LSB



Current Status of LSB Development

- Formal and informal partnership with more than seven companies
- Prototypes from 8 kW (2.5 cm i.d.) to 10 MW (30 cm i.d.) all with ultra-low NO_x capability
- Demonstrated 60:1 turndown
- Demonstrated multi-fuel capability (pure hydrogen and other fuel blends)
- Tested for process heating, domestic water heater, pool heater, small boilers, and refinery gas boilers
- DOE-OIT funding to collaborate with boiler and burner OEM's
- Licenses available for boiler applications

Transferring Low-Swirl Combustion to Gas Turbines

- Stationary land turbines are first to adapt lean premixed combustion to reduce NO_x
 - Dry Low NO_x (DLN) technology eliminates the need for steam injection
 - Current products guarantee $< 25 \text{ ppm NO}_x$ (@ 15% O_2)
- New air-quality regulations in California and many parts of US require $< 5 \text{ ppm NO}_x$ (@15% O_2) by 2005
 - DOE supporting research projects on catalytic combustors and surface stabilized combustors
 - ♦ Expensive materials that may degrade overtime
 - ♦ Elaborate controls needed to maintain smooth operation

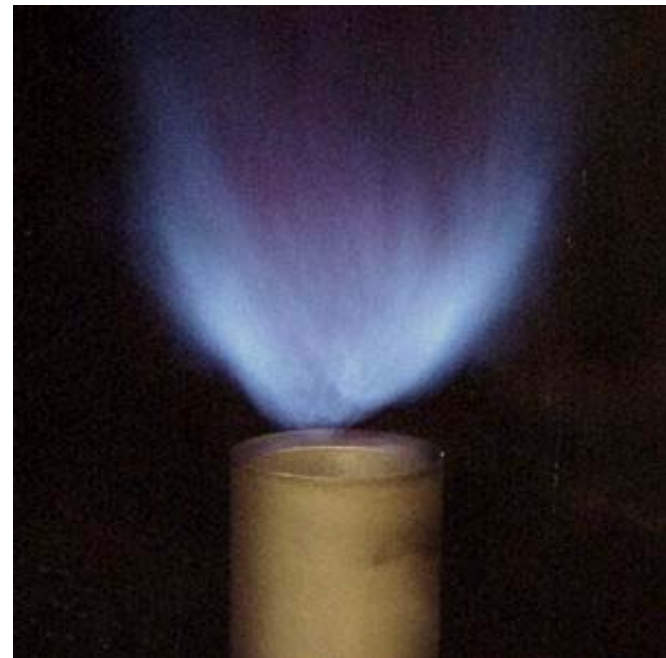
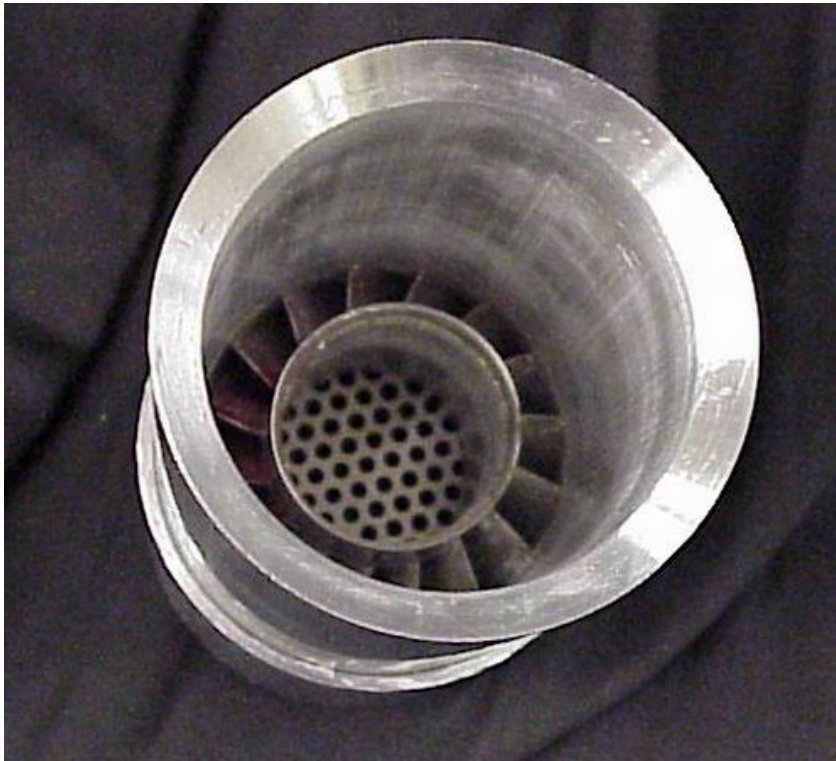
Current DLN Gas Turbine Engines Use High-Swirl Injectors

- Center bluff body promote formation of recirculation
- Flame attachment at centerbody rim



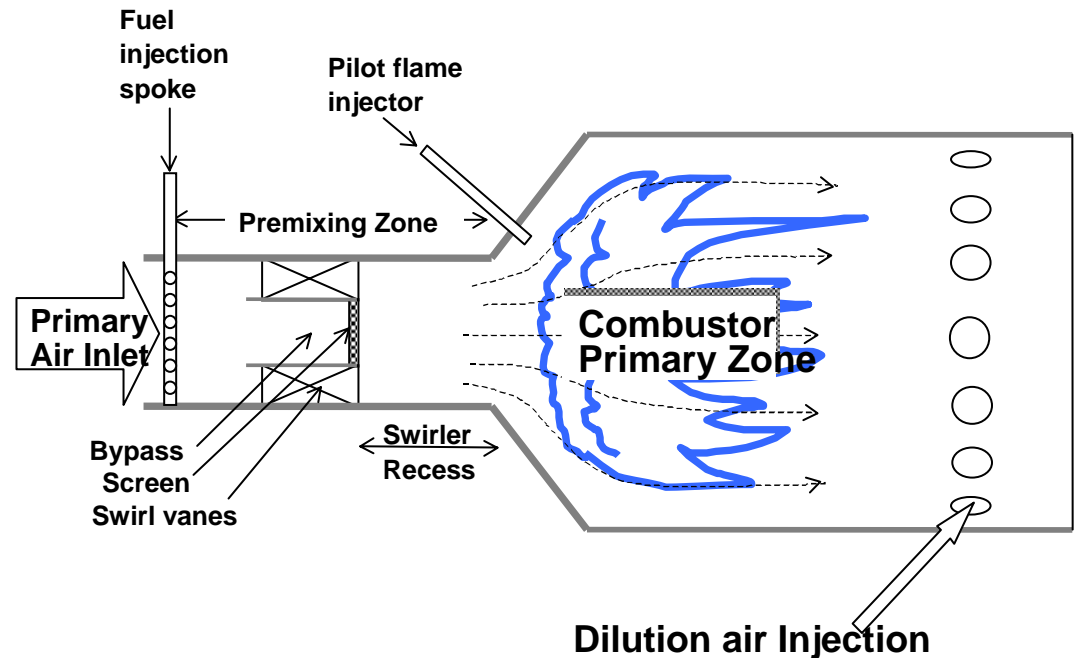
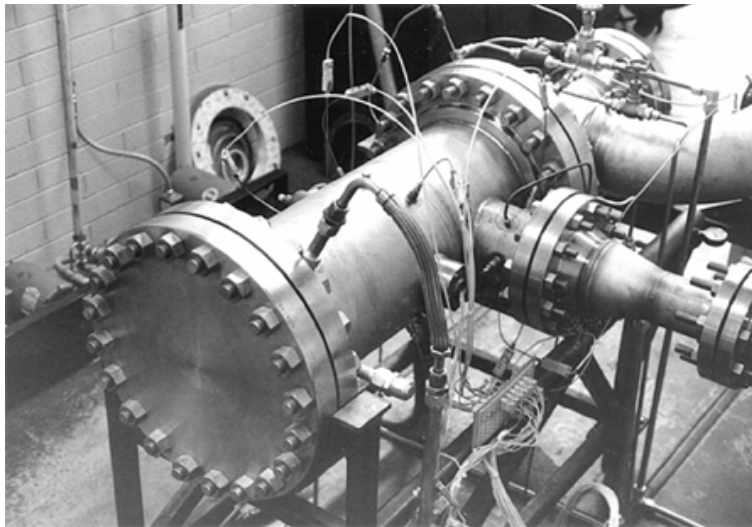
Low-Swirl Injectors (LSI) Configured in the Laboratory

- ❑ Removed centerbody from SoLoNOx swirler
- ❑ Fitted with an exit tube (using 1.5 D rule)
- ❑ Vary center channel screen to optimize flame lift off height at 6-8 m/s



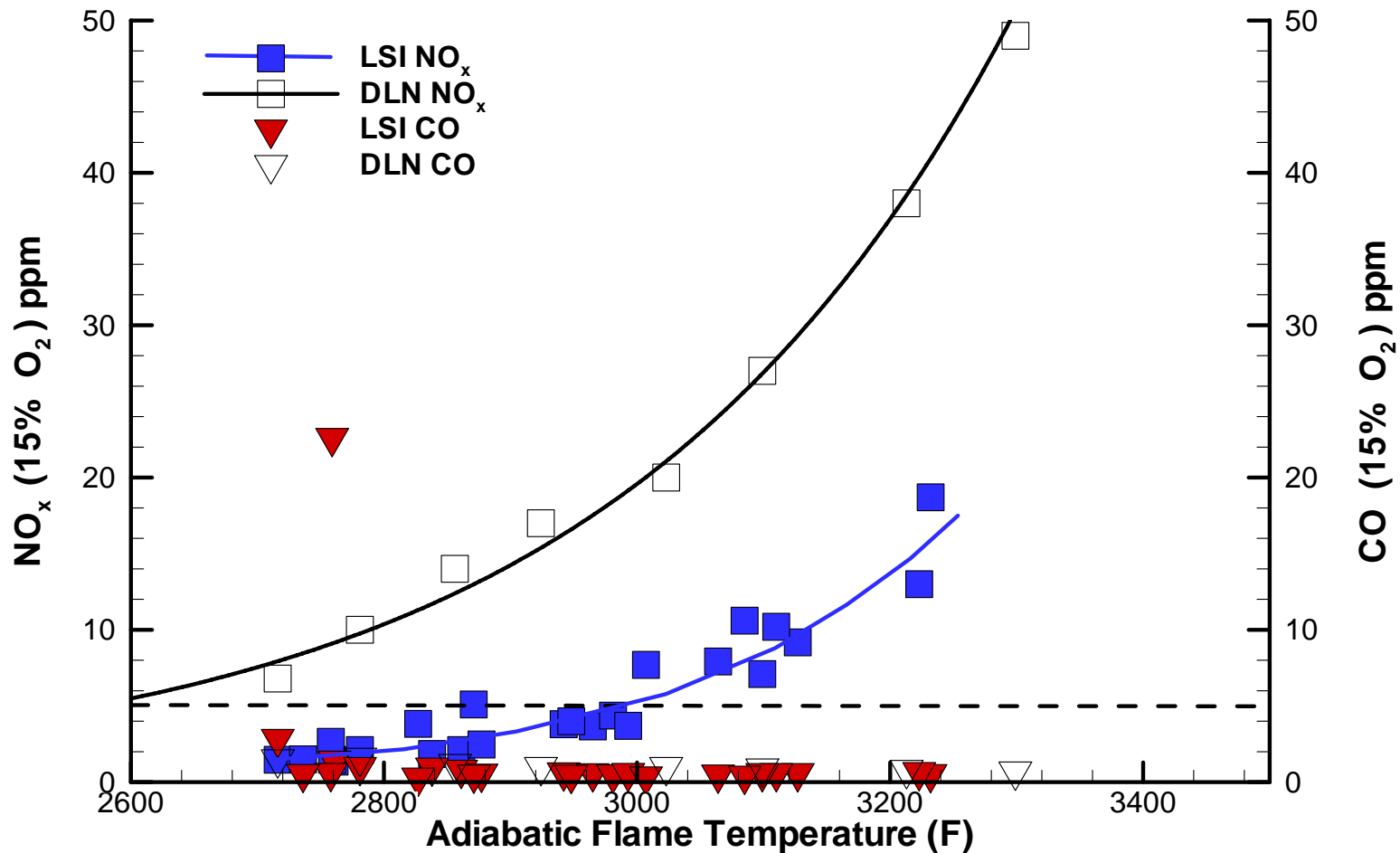
LSI Evaluated in a High Pressure Test Rig

- Mounted LSI prototype in a Solar louvered combustor liner
- Tested at up to 600C preheat and at 15 atm
- Fuel introduced through fuel spokes or Solar's advanced premixer

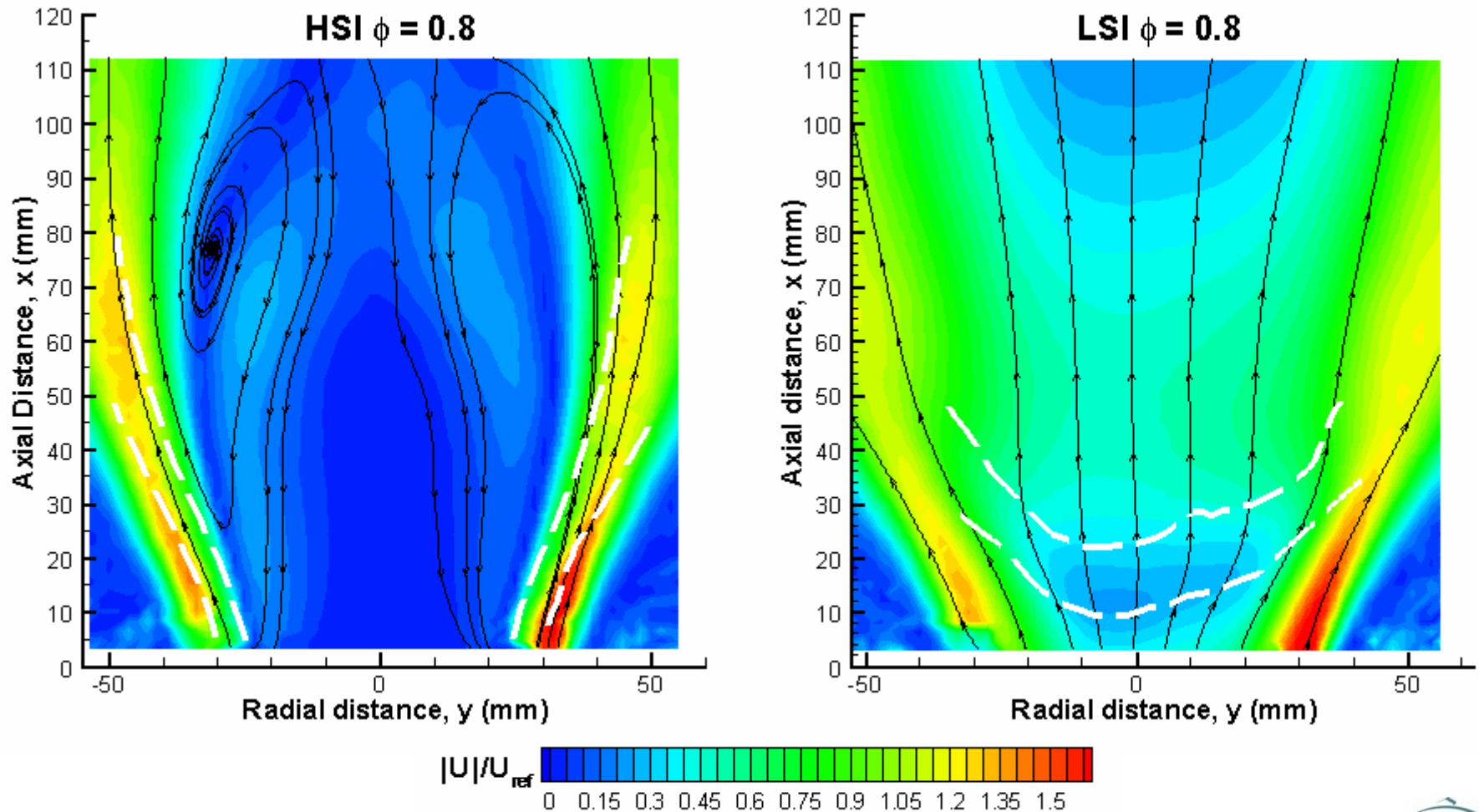


LSI Achieved < 5 ppm NO_x (15% O₂) at Full Engine Load Conditions

- Over 60% NO_x reduction compared to HSI with no compromise on performance



Absence of Large Recirculation in LSI Explains NO_x Reduction



Demonstrated LSI Concept for Mid-size Gas Turbine

- Fully compatible with existing engines
 - LSI prototypes made from SoLoNO_x production hardware
 - very low add-on cost expected for implementation
- Lowest emissions matching those of catalytic combustors
 - No compromise on duty cycle time, and a much less elaborate and lower cost alternative
 - NO_x < 5 ppm conditions far from LBO & oscillations
- Show good promise to maintain low emission under partial load
 - does not required staging to maintain low emissions under partial load

Fundamental Research Enriched by Technology Transfer

□ Provides important insights

- Strong correlation of NO_x emissions with flame temperature for systems from 7 kW to 3MW
- High combustion efficiency despite intense turbulence

□ Identifies research needs

- Turbulent displacement flame speed at high velocities
- Flame properties at high inlet temperatures and pressures

□ Indicates knowledge gaps

- Fundamental properties of flames burning with multi-component and low heating value fuels
- Burner/chamber coupling and post combustion chemistry and modeling

Planned RD&D Activities

□ LSB

- Process heat – develop enhancement methods with Maxon: staging, internal FGR and preheat
- Boilers & petroleum refining – continue testing with potential development and commercialization partners

□ LSI

- Mid-size turbines – begin engine test in Winter 2004
- Micro & utility turbines – seeking research & development partnerships and opportunities

□ Enabling technologies

- Partial reforming – seeking demonstration partners
- Alternate fuels – demonstrated firing with H_2 , HC/H_2 , biomass & low-Btu fuels. Seeking R&D opportunities
- Prevaporized premixed liquid fuels – initiated research at Nat'l Aerospace Lab. of Japan and discussion with U of Wash.
- Combine heat & power generation – LSB+LSI: seeking R&D opportunities

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